

A STUDY OF THE DISPERSION ON PRIMARY CALIBRATION RESULTS OF SINGLE-ENDED ACCELEROMETERS AT HIGH FREQUENCIES

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Abstract: This paper describes a study developed at INMETRO, which is focused on the evaluation of primary calibrations of single-ended accelerometers at high frequencies. It proposes a simple solution to reduce the dispersion on the sensitivity results evidenced when different vibration exciters are used. This can be helpful for calibration laboratories to compare results obtained in different systems. Therefore, the authors consider that it might benefit the process of both systems validations and interlaboratory comparisons.

Keywords: primary calibration, vibration, accelerometer, metrology, interferometry.

1. INTRODUCTION

According to the international standard ISO 16063-11 [1], primary laser interferometric calibration of accelerometers is based on the determination of the electrical output of the device under test to a given acceleration input at its reference surface. In the case of single-ended accelerometers, the interferometric measurement of the input motion quantity is made on the mounting surface close to the accelerometer base.

In the first CIPM vibration key comparison CCAUV.V-K1, two accelerometers, one back-to-back and one single-ended, were circulated for primary calibration between 40 Hz and 5 kHz. Unfortunately, no key comparison reference value was determined for the single-ended accelerometer because a large dispersion was observed on the results reported by the participants [2].

Before the start of the currently ongoing key comparison CCAUV.V-K2 [3], the pilot laboratory (PTB/Germany) needed to change their high frequency vibration exciter. During this process they evidenced some differences between the sensitivities determined using their new exciter in comparison to the older one. This has been reported by Dr. Thomas Bruns in 2009, during the IMEKO TC 22 technical meeting in Lisbon. Later on, two articles related to this problem were published in *Metrologia* [4][5].

This behavior has also been observed at the vibration laboratory of INMETRO during the validation stage of our primary quadrature calibration system, when different vibration exciters were tested. The problem has been

analyzed and a research was carried out aiming to reduce the systematic deviation observed at high frequencies.

A proposal to minimize this effect is presented herein, which can certainly benefit other laboratories that want to carry out internal validations through comparisons between different calibration systems. It can also be helpful to future interlaboratory key and supplementary comparisons in vibrations.

2. DISPERSION OBSERVED IN DIFFERENT EXCITERS

Bruns *et al.* [4] have reported the comparison of charge sensitivity results of a single-ended (SE) accelerometer B&K 8305-001 on two different vibration exciters: a PTB modified version of the Bouche shaker (VE) and a SPEKTRA SE-09 shaker. The results obtained exhibited a systematic deviation which increased with rising frequency, starting at 5 kHz. Relative deviations up to 2 % at 10 kHz and 4 % at 20 kHz were evidenced. It was also observed that instabilities of this order could have drawn the use of the SE artifact for the key comparison CCAUV.V-K2 into question.

According to this PTB study the cause of this deviation can be a material dependency of the measured magnitude of sensitivity. The VE and SE-09 shakers analyzed in their study use armatures made of beryllium and ceramics, respectively. Probable causes of this effect can be either differences in contact stiffness or modal deformation of the armature, where the acceleration input is interferometrically measured. This was justified by results of optical measurements onto the base of a “naked” accelerometer (without case) and onto the shaker armature close to the accelerometer. FEM analysis and scanning laser vibrometry measurements onto the surface of the armatures have shown radial relative acceleration deviations. This proves that the armatures are not rigid bodies as they are normally assumed to be.

Bruns *et al.* have considered in a second publication [5] the inclusion of additional spring and damping elements in the mathematical modeling for a primary calibration of an SE accelerometer, in order to consider a finite contact stiffness between the accelerometer and the armature. It was concluded that there are strong evidences that the cause of the problem is a non-negligible relative motion between the SE transducer and the shaker armature at high frequencies.

These studies carried out at PTB were restricted to calibration shakers models, which include lightweight armatures of high stiffness materials. Most of these calibration exciters use air-borne bearings and present limited maximum force ratings. Other National Metrology Institutes (NMI) have different models of shakers in use, which can produce higher levels of deviations. These shakers are usually older models not initially designed for primary calibration purposes, which can be preferred in some specific conditions. For example, this might be the case when higher acceleration levels are required at high frequencies, when the fringe disappearance or minimum point methods are used [1].

The problem regarding low stiffness of the armature was already presented by the authors of this paper in a former publication [6]. It was shown that the charge sensitivity values obtained for a standard single-ended accelerometer mounted on three different vibration exciters presented large deviations above 1 kHz due to modal deformations. The results obtained at 3 kHz for two aluminium- armature shakers were much higher than the results obtained with an air-bearing guided beryllium-armature model. The authors have also introduced that using a high stiffness adapter it was possible to get equivalent results using aluminium and beryllium made armatures.

3. MATHEMATICAL MODEL AND SIMULATIONS

Piezoelectric accelerometers can be represented as a lightly damped single degree-of-freedom spring-mass system. In order to simplify the analysis, it will be considered here that the accelerometer is ideally mounted with infinite stiffness to a rigid body type armature. In this case, accelerometers can be modelled by the classical second order differential equation whose solution is

$$S(f) = \frac{S_0}{\sqrt{\left(1 - \left(\frac{f}{f_n}\right)^2\right)^2 + 4\zeta^2 \left(\frac{f}{f_n}\right)^2}} \quad (1)$$

$$\Delta\varphi(f) = -\arctan\left(\frac{2\zeta\left(\frac{f}{f_n}\right)}{1 - \left(\frac{f}{f_n}\right)^2}\right) \quad (2)$$

where:

- f_n is the undamped natural frequency in Hz,
- f is frequency in Hz,
- $S(f)$ is the sensitivity magnitude at any frequency,
- S_0 is the low frequency reference sensitivity magnitude,
- $\Delta\varphi(f)$ is the sensitivity phase shift at any frequency,
- ζ is the damping factor.

The variation of sensitivity against frequency of an ideally mounted reference accelerometer is given by equations (1) and (2). If a high stiffness mechanical connection between the accelerometer and the armature is

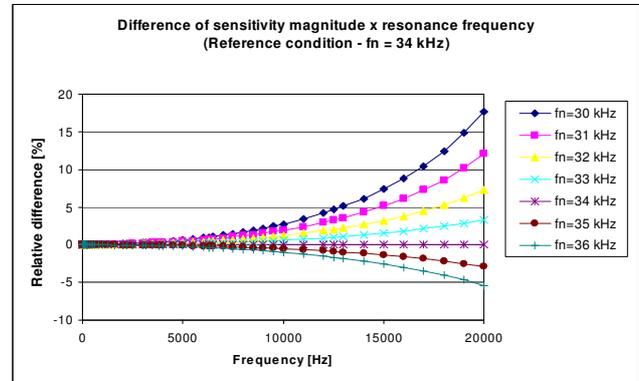
not achieved, a reduced mounted resonance frequency might be observed in the measured frequency response function. In this case, an abnormal increase of the accelerometer sensitivity occurs at high frequencies and significant effects can be seen above 5 kHz.

Most common accelerometers and reference accelerometers have resonances in the 20 kHz to 60 kHz range. Table 1 presents the nominal mounted resonance frequencies of single-ended reference accelerometers from different manufacturers. For practical use and for the simulations presented in sequence, f_n will be approximated by the accelerometer mounted resonance frequency.

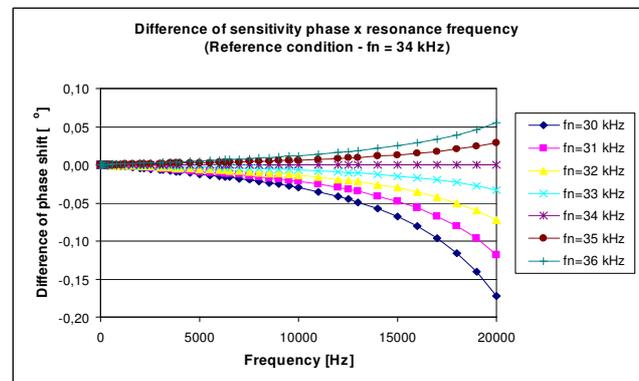
Table 1 – Nominal mounted resonance frequencies of SE reference accelerometers obtained from their technical specifications

Manufacturer	model	Base material	Frequency
Brüel & Kjaer	8305-001	316 stainless steel	34 kHz
Endevco	2270 M8	17-4 PH stainless steel	55 kHz
Kistler	8002 K	303 stainless steel	40 kHz

In order to evaluate the effect of the change in f_n relative to the nominal value specified by the manufacturer, some simulations were performed using eq. (1) and (2). Measurements carried out at INMETRO have shown that the damping factor of an accelerometer model B&K 8305-001 is approximately $\zeta = 5 \times 10^{-3}$. This value will be used here for the simulations.



(a)



(b)

Fig. 1 – Difference of the sensitivity (a) magnitude and (b) phase of a SE accelerometer from a reference mounted resonance frequency of 34 kHz.

Fig. 1 presents simulation results considering an accelerometer B&K 8305-001. It shows the difference between sensitivity values obtained for different mounted resonance frequencies and the values relative to a nominal mounted resonance frequency of 34 kHz.

It can be seen that a change in the resonance mounted frequency of a SE accelerometer affects considerably its sensitivity response. If the resonance mounted frequency is reduced, occurs a sensitivity magnitude increase and a phase shift reduction. This effect increases with the vibration frequency. For example, in the case of a resonance mounted frequency 5 % lower than 34 kHz, the sensitivity magnitude is increased by 1 % at 10 kHz and 5,7 % at 20 kHz. The influence on phase shift is quite smaller, being approximately 0,01° lowered at 10 kHz and 0,06° at 20 kHz.

4. MOUNTING SURFACE CONDITIONS

As already mentioned, one important factor that affects the mounting stiffness of an accelerometer is the quality of the matting surfaces. It is important to assure that both the SE accelerometer base and the surface which it will be attached to are free of scratches and present proper flatness and roughness. The use of a proper lubricant as light oil or silicon grease helps to assure a high mounting stiffness and obtain the highest possible mounted resonance frequency.

During the torque application to stud mounted accelerometers it is usual to appear circular scratches on its base. This might cause an increasing degradation of the mounting quality condition with time. One possible way to overcome this problem is to submit the accelerometer to re-lapping in order to recover its original condition.

Fig. 2 shows the effect of the quality of mounting base on the sensitivity of a SE accelerometer measured by a primary laser interferometric system at INMETRO. This figure includes both the measurement results obtained with the accelerometer as received from a customer and after a re-lapping of its base was performed. The increase of the mounted resonance after improvement of the mounting surface condition is clearly noted in frequencies above 5 kHz. The reduction on the measured sensitivity in this case was 1,1 % at 10 kHz and 4,5 % at 20 kHz.

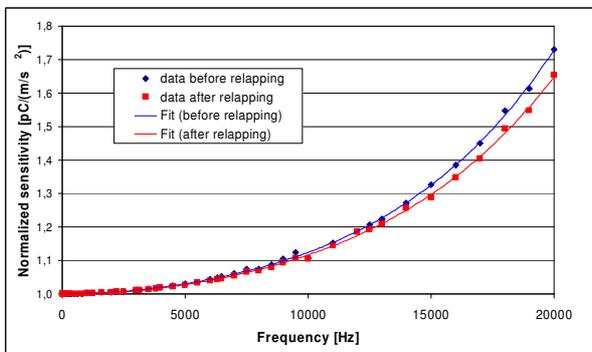


Fig. 2 – Difference of the sensitivity magnitude measured before and after relapping the base of a SE accelerometer.

During a typical vibration interlaboratory key comparison [3] it is usual to have more than ten

participating laboratories performing around five repeated series of calibrations. Considering that each series includes two mountings to minimize the coupling effect between transverse accelerations generated by the vibration exciter and the transverse sensitivity of the accelerometer, this would require a minimum of fifty mountings. If in addition ten preliminary measurements are carried out by the pilot laboratory to access the stability of the artifact and another ten monitoring measurements are carried out along the circulation period, it easy to reach a number over one hundred mountings. This high number of mountings might require several re-lappings of the accelerometer base during the period of the comparison in order to assure that every participant receives the DUT with approximately the same mechanical conditions.

However, the re-lapping process requires special skills, it might pose risks to the integrity of the artifact and is based on the removal of material. If several re-lappings are necessary, some change in the sensitivity can be expected.

If the number of mountings is reduced considerably, the re-lapping process can be eliminated and the accelerometer would probably maintain its original characteristics with minimum risks.

5. PROPOSAL OF ADAPTER

A very simple solution has been used at INMETRO to reduce the number of mountings and the dispersion between the results obtained using different vibration exciters. The accelerometer is stud mounted onto the top surface of a mechanical adapter, which is then fixed to the armature of the different shakers available in our vibration laboratory. This adapter can be attached directly to the shaker armature or via a secondary adapter to fit it to the different fixture patterns. These additional adapters can also be used to allow rotation of the set at specific angles.

Fig. 3 shows the mechanical design of an ANSI 304 stainless steel adapter. Its top surface is polished for direct optical reflection of the laser beam. It includes a # 10-32 UNF threaded through hole for stud mounting of the accelerometer and a transverse 3 mm diameter hole to help providing the required mounting torque. The 17mm diameter neck is to reduce mass and allow the mounting on the armature of a B&K calibration head type 4815.

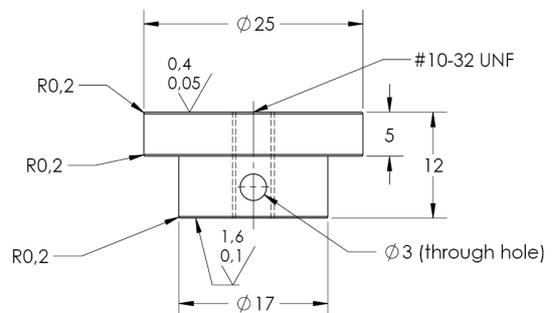


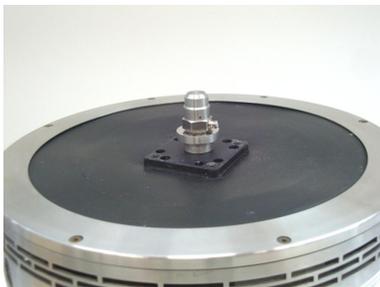
Fig. 3 – Mechanical design of the mounting adapter for SE accelerometers

Fig. 4 shows a single-ended reference accelerometer B&K 8305-001 mounted on the top surface of this stainless steel adaptor. The red lines illustrate laser beams reflecting onto its polished top reference surface.



Fig. 4 – SE reference accelerometer mounted on top of the stainless steel mounting adapter

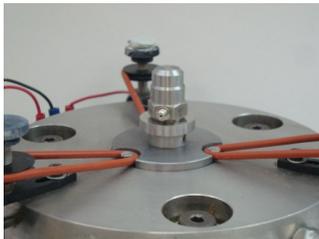
This adapter allows direct mounting to armatures of many different vibration exciters. This is exemplified in Fig. 5, which shows the accelerometer plus adapter set mounted on the following models: (a) B&K high g shaker head model 4811; (b) B&K shaker model 4809; (c) Endevco model 2901; and (d) PCB shaker model 396C10.



(a)



(b)



(c)



(d)

Fig. 5 – SE reference accelerometer and adapter mounted on top of the moving element of different vibration exciters: (a) BK 4805 + BK 4811 high g head; (b) BK 4809; (c) Endevco 2901; (d) PCB 396C10.

An additional adapter might be necessary to mount the system shown in Fig. 4 to some exciters like the B&K calibration head model 4815, which presents a recessed mounting plane. Fig. 6 (a) presents a square shaped aluminium adapter to allow proper fitting to the mounting hole pattern of this shaker and to raise its mounting plane. This aluminium adapter also allows rotation of the set in 90° angle increments. The final mounting of the set onto the shaker armature is shown in Fig. 6 (b).



(a)



(b)

Fig. 6 – SE reference accelerometer and adapter mounted on top of the moving element of a vibration exciter BK 4805 + BK 4815 calibration head

It should be noted that despite the lower stiffness of this in-series aluminium adapter, there is negligible influence on the calibration results. The original shaker armature and adapter provided by B&K are also made of the same material and its low weight helps to reduce the total mass loading mounted on the top of exciter.

FEM analyses have shown that the proposed adapter presents resonance frequencies far beyond the frequency range of interest. The two lower resonance frequencies refer to tilting modes at 30 kHz and to a rotational mode at 32 kHz, approximately. The first flexural mode of the top reference surface, which occurs above 58 kHz is shown in Fig. 7. This flexural deflection mode would very probably introduce errors on the acceleration determined by primary laser interferometric methods and could additionally induce stress in the piezoelectric discs inside the accelerometer under test.

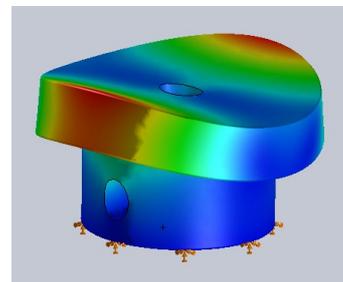


Fig. 7 – Flexural mode at 58317 Hz.

6. MEASUREMENT RESULTS

In order to exemplify the application of the proposed adapter, and to illustrate the level of improvement achieved, some experimental results will be presented herein.

The frequency response of a SE standard accelerometer B&K 8305-001 mounted with the stainless steel adapter on the beryllium moving element of a vibration exciter Endevco 2901 is shown in Fig. 8. The acceleration measurements were performed at two diametrically opposed points (M1P1 and M1P2) on the surface of the adapter by a laser vibrometer Polytec model VDD-660. The calculated mean sensitivity curve (Mean M1) presents a peak value at approximately 33,5 kHz, which is only 1,47 % lower than the nominal mounted resonance frequency given in the technical specifications of the accelerometer.

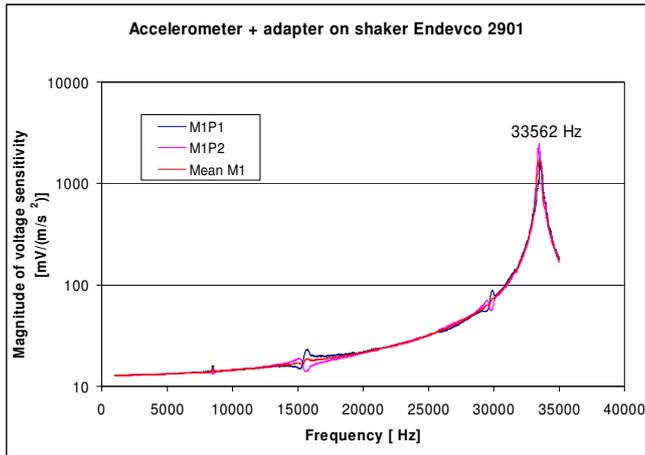


Fig. 8 – FRF of an accelerometer + adapter mounted on the moving element of a vibration exciter Endevco 2901

This level of agreement provides an evidence of the good quality of the mounting stiffness. The knowledge of the mounted resonance frequency allows the calculation of the magnitude and phase responses of an accelerometer using equations (1) and (2) respectively for comparison with the experimentally determined sensitivities.

Fig. 9 presents a comparison between the results of charge sensitivity measured using a flat-spring-suspension exciter model B&K 4815 against the results obtained using an air-bearing guided beryllium-armature, exciter model PCB 396C10. The sensitivities presented are normalized by the reference value measured at 160 Hz. These measurements were carried out with the B&K exciter using the adapters shown in Fig. 6(a) and with the PCB shaker with and also without the stainless steel adapter.

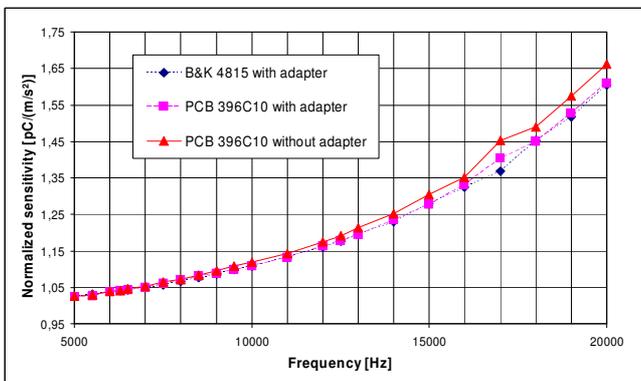


Fig. 9 – Normalized sensitivity results obtained using an exciter BK 4815 with adapter and an exciter PCB 396C10 with and without adapter.

The results obtained using the accelerometer mounted directly onto the beryllium armature of the PCB shaker present a systematic increase of the sensitivity at high frequencies. Deviations of 0,9 % and 3,5 % are observed at 10 kHz and 20 kHz, respectively. Using the stainless steel adapter on both shakers, a very good agreement is obtained. The deviations at these same frequencies are then reduced to 0,04 % and 0,3 %. It should be noted that the residual effect on the sensitivity seen at 17 kHz is caused by an internal resonance of the accelerometer. Apart from this frequency, very smooth curves are obtained.

A similar behaviour has also been observed when using a vibration exciter Endevco model 2901 both with and without the stainless steel adapter. This other shaker also includes an air bearing guided beryllium-armature.

The reduction in the dispersion at high frequencies is evidenced in Fig. 10, which presents the relative difference between sensitivity magnitude results obtained using three different shakers with and without the adapter proposed. The relative values are calculated in reference to the sensitivity results obtained employing a shaker B&K 4815 with the mounting configuration shown in Fig. 6. In order to facilitate visualization of the improvement achieved, this graph includes polynomial curves fitted to the measured data and excludes some few spurious frequencies contaminated by internal resonances of the accelerometer. The agreement between the sensitivity magnitude results obtained using the adapter on these three different shakers is within $\pm 0,3$ % between 1 kHz and 10 kHz, excluding 5,5 kHz where a deviation of 0,4 % is observed.

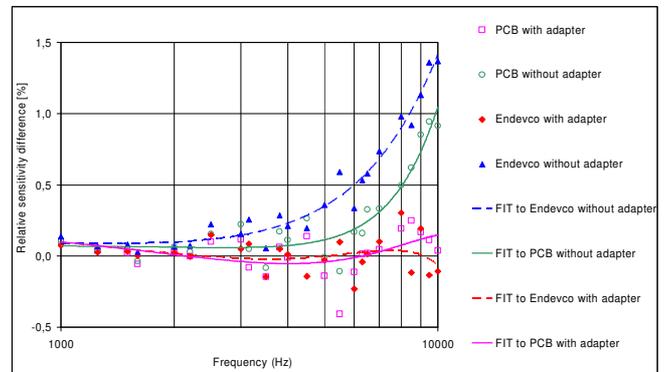


Fig. 10 – Difference of sensitivity results using exciters Endevco 2901 and PCB 396C10 relative to the ones obtained using a B&K 4815 with adapter.

7. CONCLUSION

A study of the dispersion on primary calibration results of SE accelerometers has been presented in this paper. It was confirmed that the results obtained at high frequencies can differ significantly depending on the vibration exciter used. This effect might have different causes, including the different mounting stiffness conditions and modal deformations on the armature of vibration exciters.

Considering the fact that SE transfer accelerometers are intended to calibrate BTB standards or calibration shakers with built-in reference accelerometers, it is important to calibrate them in the same condition or as similar as possible condition to its end use.

Currently, exciters with armatures made of aluminium, beryllium, and ceramics are available at NMIs for use in calibration of accelerometers. This variability of mounting conditions might yield in different mounted resonance frequencies, which then affects the sensitivity results obtained by each laboratory at high frequencies.

This paper focuses on a proposal that can minimize the dispersion problem. It was shown that the use of a stainless steel mounting adapter with high contact stiffness properties and no vibration modes within the frequency range of interest can improve considerably the agreement between the results obtained. Some experimental results have been presented to demonstrate that substantial improvement is obtained and the dispersion can be significantly reduced. This reduction of the effect of the vibration exciter can help laboratories to obtain better comparability between different calibration systems. This is an important issue during the stage of validation of different calibration systems or even calibration methods.

8. FINAL REMARKS

An important impact of the use of an adapter can be related to primary interlaboratory comparisons. If future comparisons consider this proposal, the dispersion caused by variation of the mounting stiffness due to the use of different exciters by the participants can be minimized. In consequence, a better agreement might be achieved and then a comparison reference value with lower uncertainty can be obtained.

Considering this, the authors will propose the inclusion of an adapter together with the accelerometers to be circulated for the future regional key comparison SIM.AUV.V-K2.

The use of a similar adapter by different laboratories could provide a better comparability between their calibration capabilities. Additionally, comparing results with and without the common adapter, would be possible to determine correction factors between the results obtained with different shaker systems.

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