

The activities of micro-force measurement below 10 mN in Center for Measurement Standards

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Outlines

- 1. CMS in ITRI
- 2. Force measurement below 1 N
- 3. Small force measurement schemes in CMS
 - 3-1. Torsion pendulum facility
 - 3-2. Flexure stage with electrostatic sensing and actuating
- 4. Summary

1. CMS in ITRI

- One of our main projects:

National Measurement Laboratory

Establish, maintain, and disseminate national measurement standards.

n **Electrical/Electromagnetic
Measurement Laboratory**

n **Quality Engineering
Department**

n **Dimensional Measurement
Laboratory**

n **Mechanical Measurement
Laboratory**

n **Dynamic Engineering
Measurement Laboratory**

n **Flow & Energy
Research Laboratory**

n **Optical Radiation Measurement
Laboratory**

n **Medical & Chemistry Research
Laboratory**

Mechanical Measurement Lab

National Measurement Standards in fields of:

Mass

Pressure

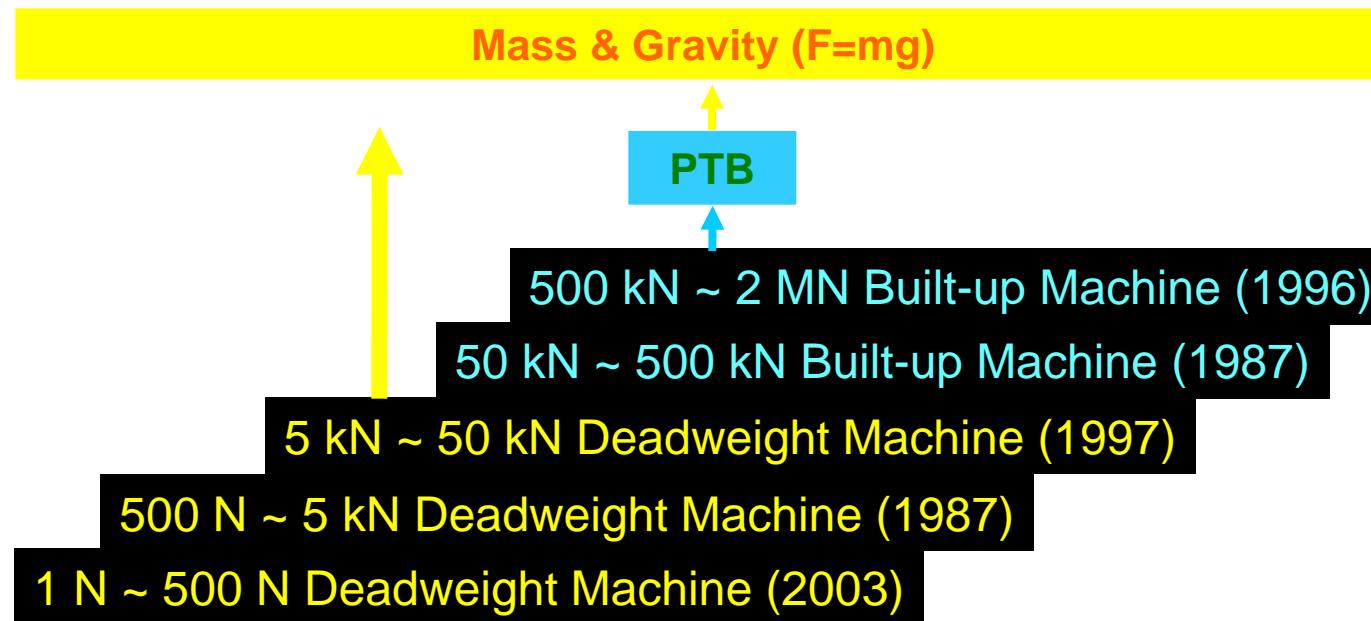


Force

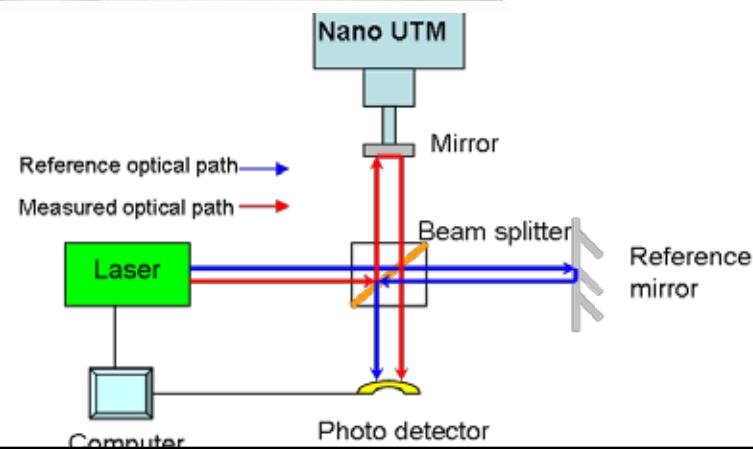
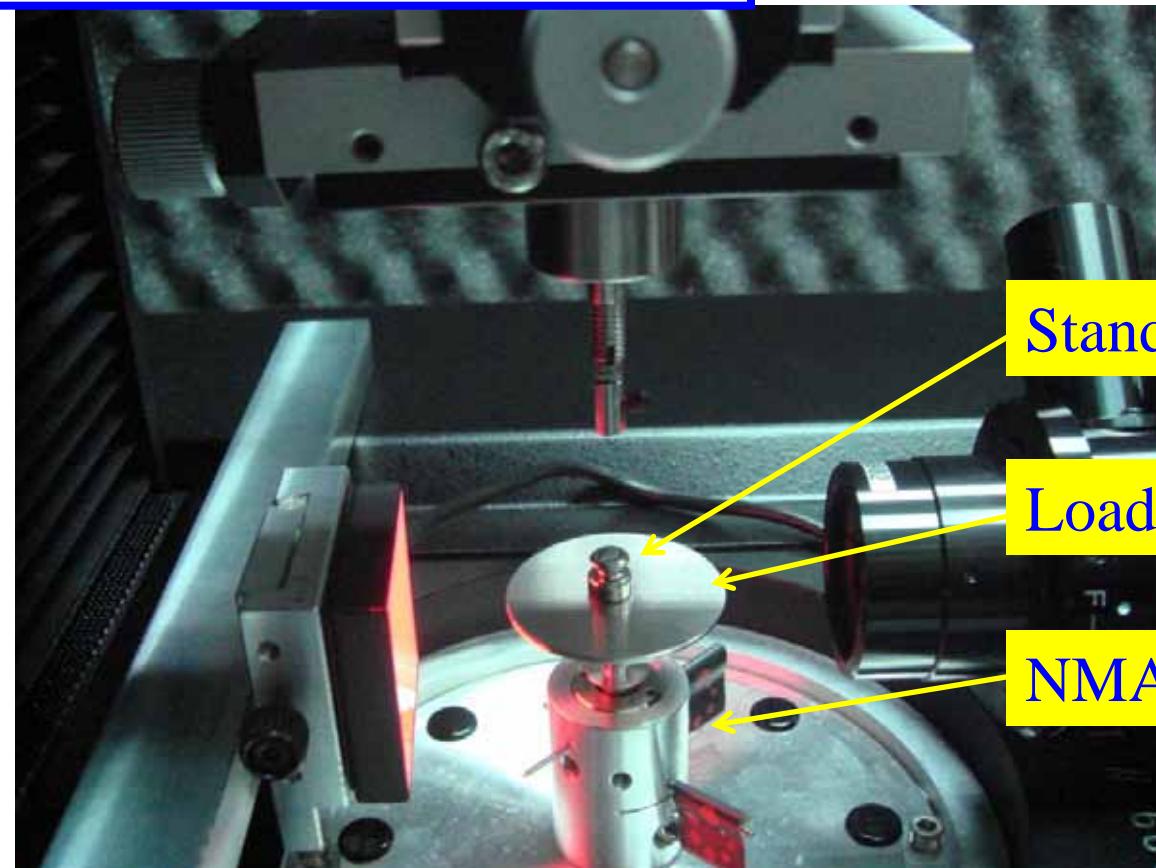
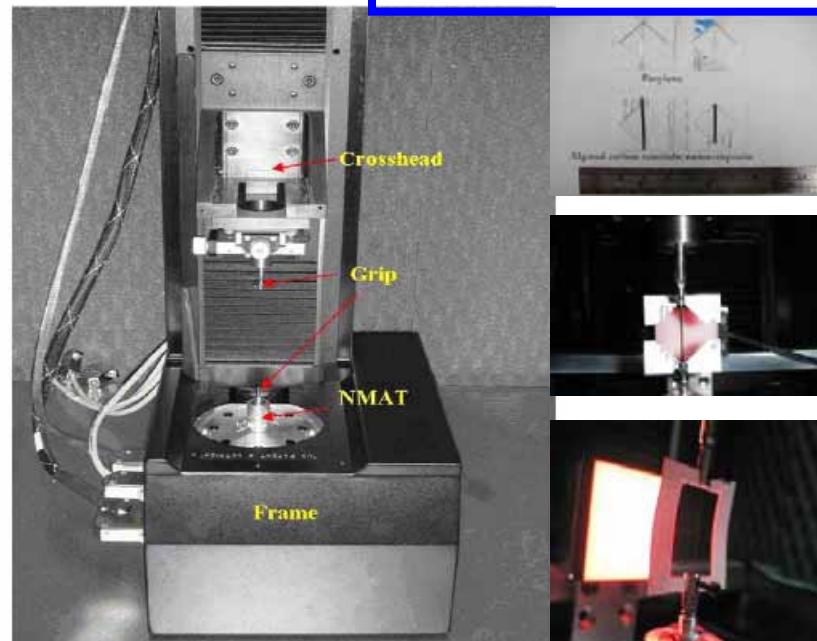
Hardness



Vacuum



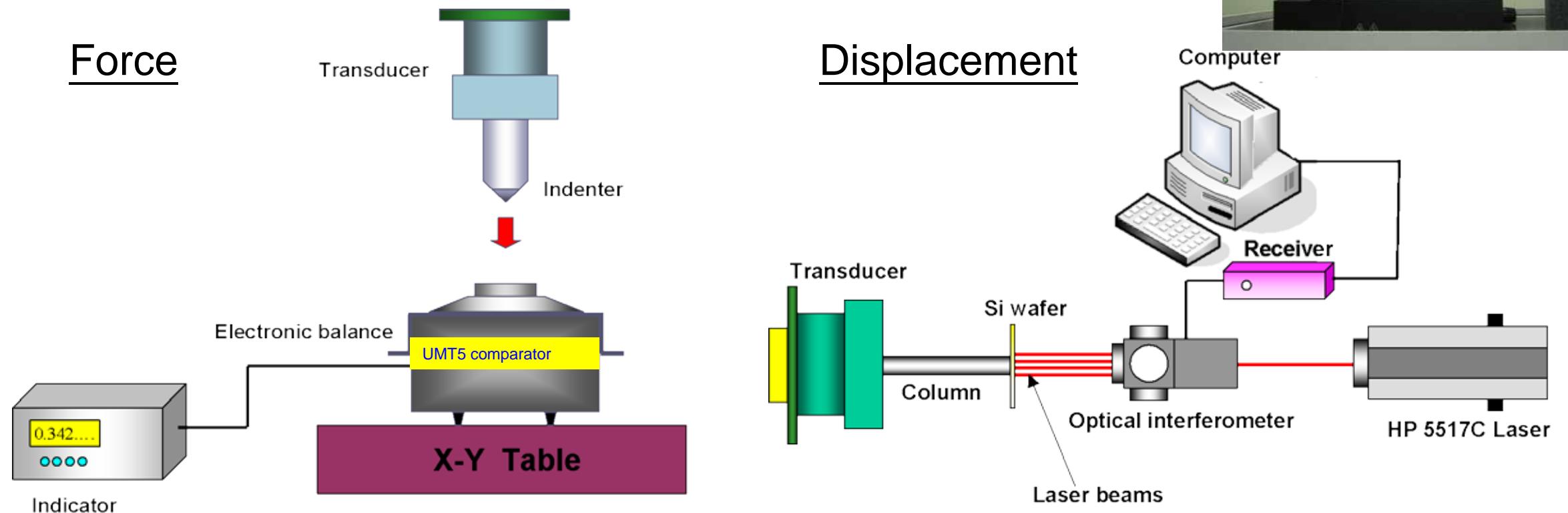
2. Force measurement below 1 N Nano Universal Testing Machine



Force range: £ 500 mN
 Extension range: £ 150 mm
 Test material: spider silk, polymer material, thin film

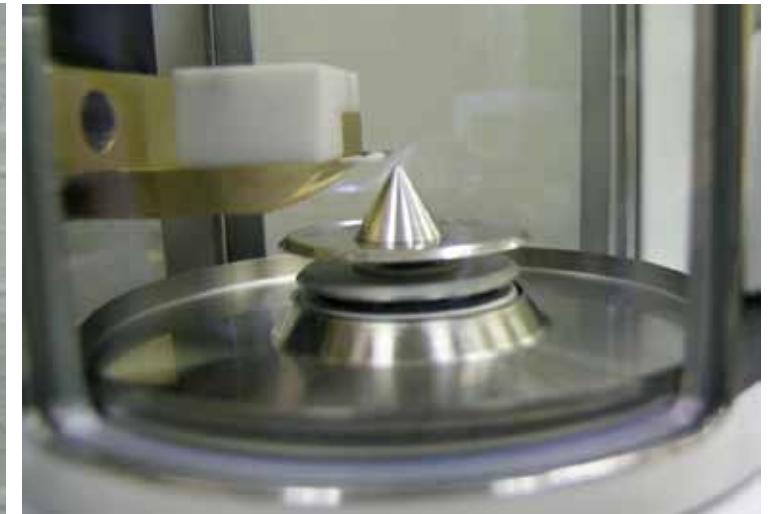
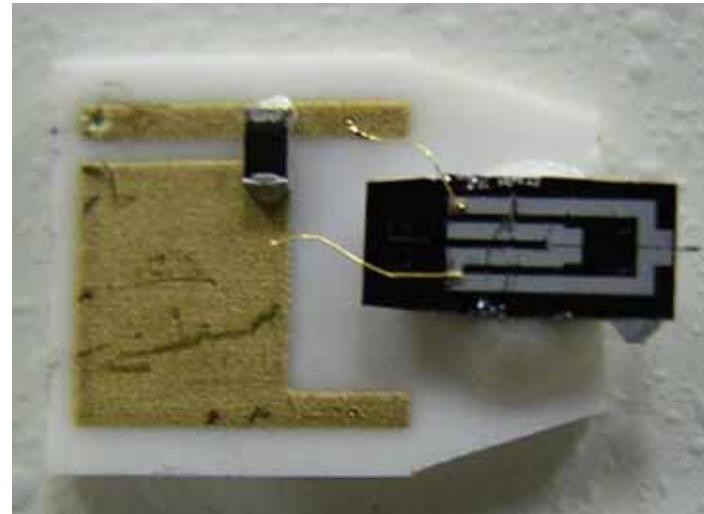
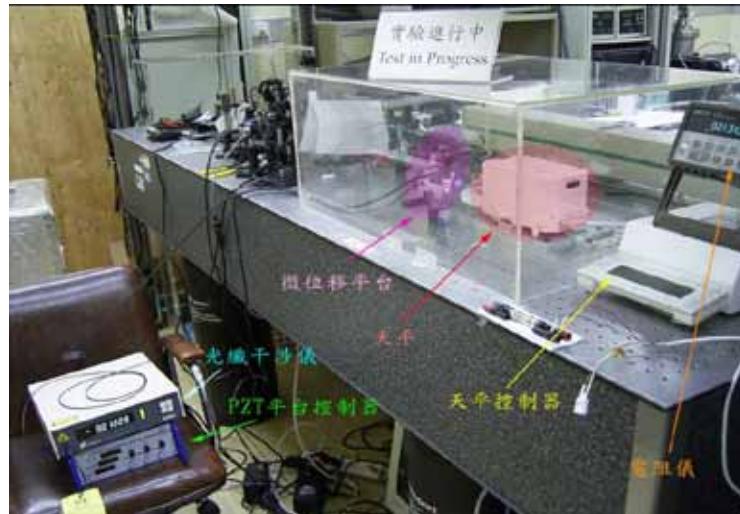
Force measurement below 1 N Calibration of Nano-indentation system

Force range: ≤ 10 mN
Indentation depth: ≤ 500 nm
Test material: Bulk material, Thin Film Specimen

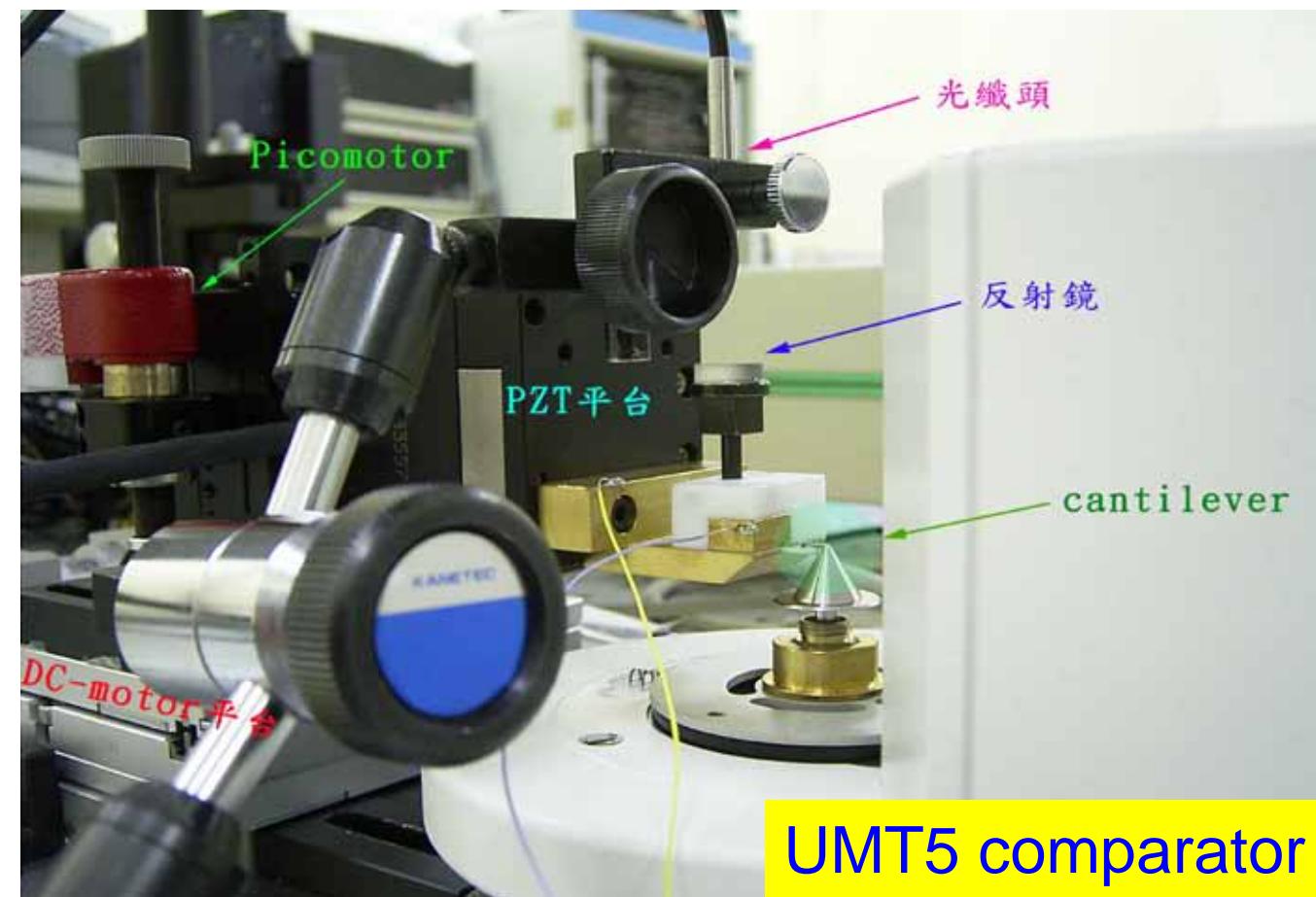


Relative expanded uncertainties with 95% confidence level:
For reduced modulus: **6.8%**
For indentation hardness: **6.3%**

Cantilever Stiffness Measurement

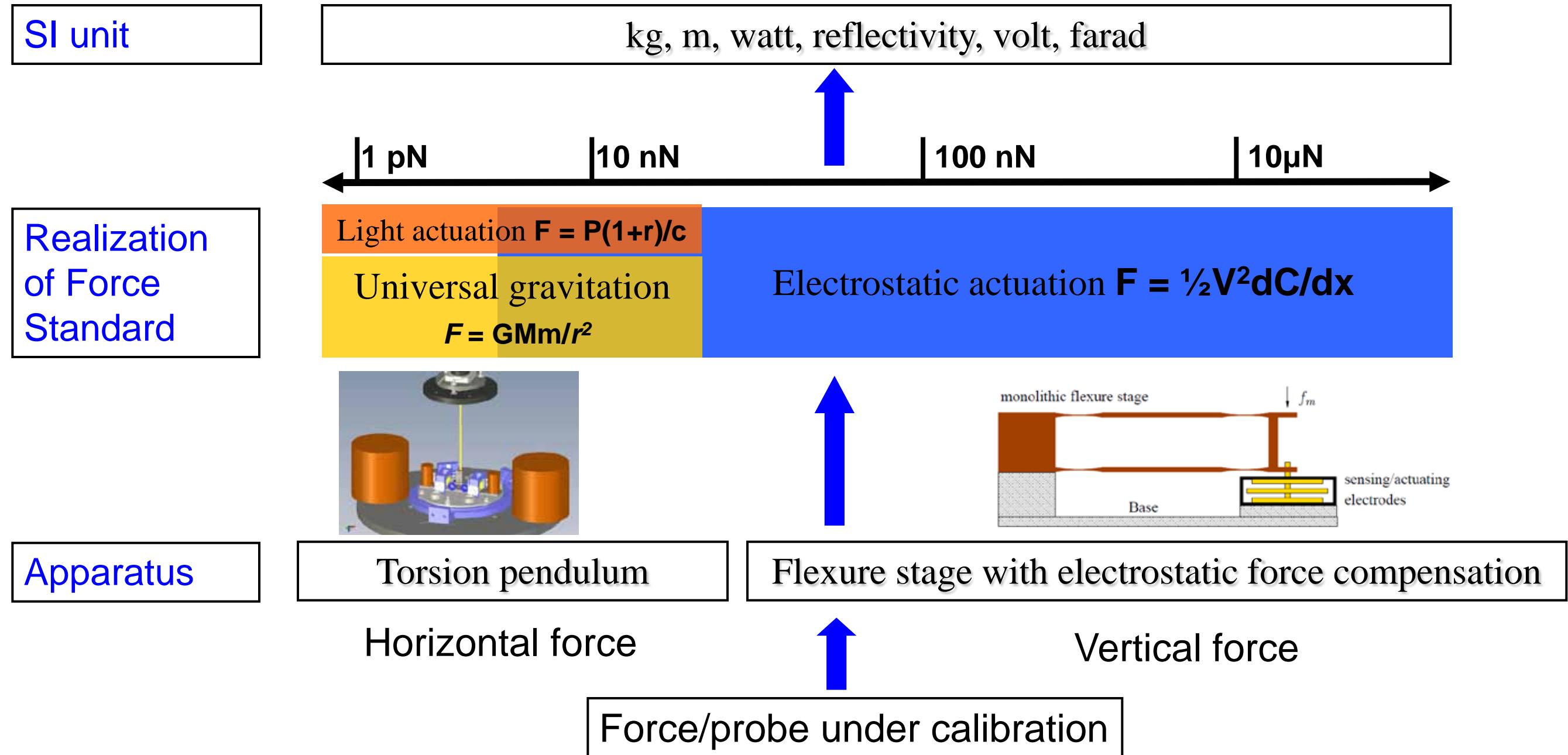


Force: below 50 mN
Stiffness: 0.5 N/m ~ 10 N/m
Uncertainty: 20%(k=2)



UMT5 comparator

3. Small force measurement schemes in CMS



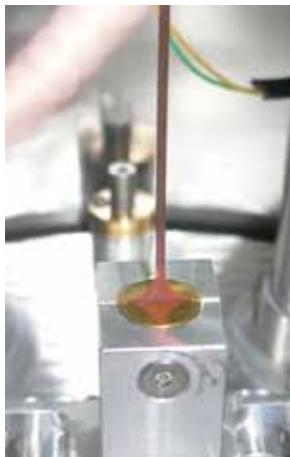
Force measurement

$$F = -k \cdot x$$

Spring constant (N/m)

Deflection (m)

Strip torsion pendulum



Flexure spring



Simple pendulum

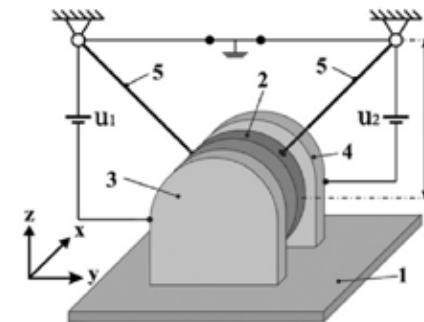


Figure 1. Electrostatic system for stiffness reduction—1: base plate; 2: conductive disc-pendulum; 3, 4: external conductive plates; 5: suspension by a thin conductive wire ($l \approx 0.3$ m).

AFM cantilever



V. Nesterov, Metrologia 46
 (2009) 277-282

Potential Resolution $dF = k \cdot dx$

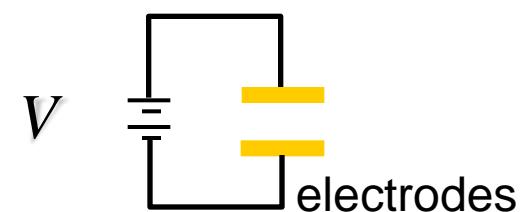
ex: $0.001 \text{ N/m} \cdot 1 \text{ nm} = 1 \text{ pN}$

Force balance measurement

- Traceable microforce generation

- Electrostatic effect

- Traceable to voltage standard provided traceable capacitance gradient measurement

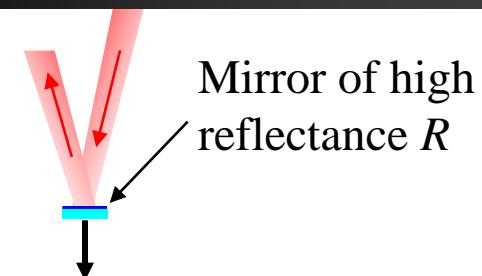


$$F_e = \frac{1}{2} V^2 \frac{dC}{dx}$$

- Radiation pressure

- Laser with stable power and frequency, plus a mirror

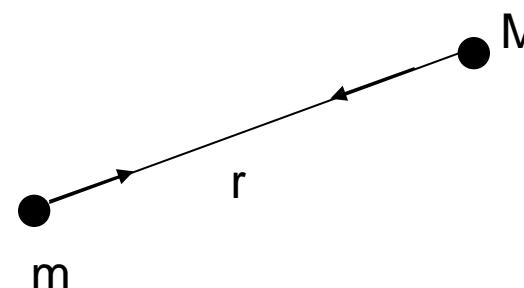
Power and frequency stabilized CW laser



$$F_r = P_r(1+R)/c$$

- Universal gravitation

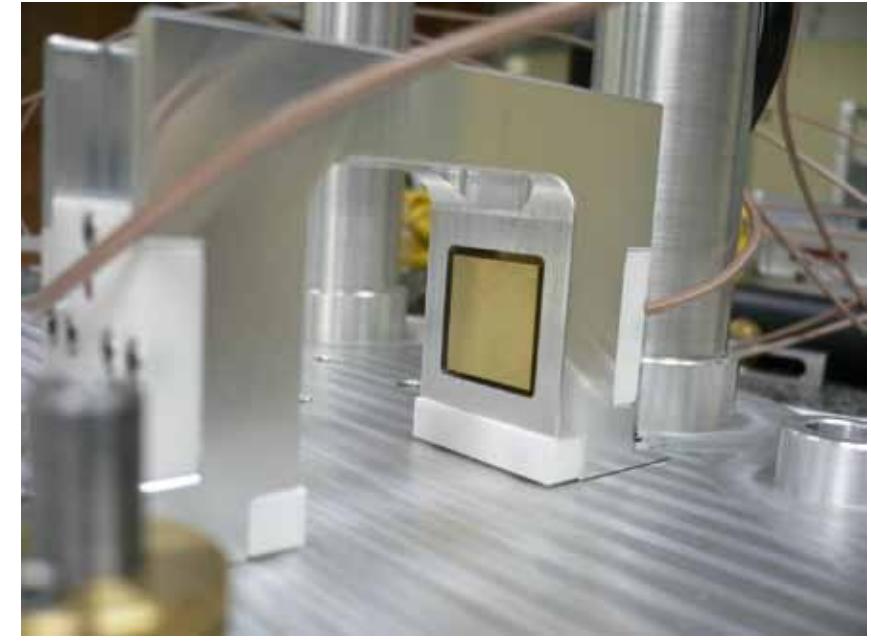
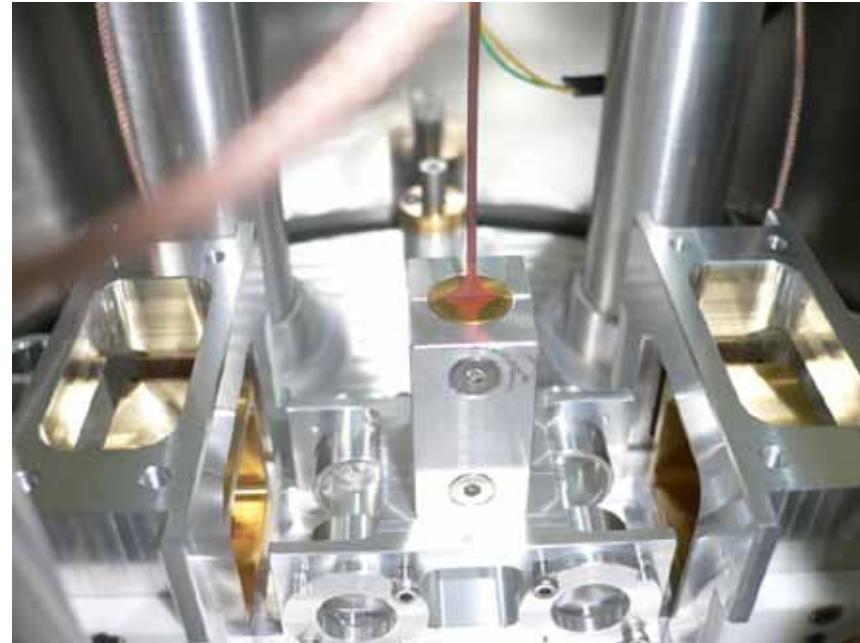
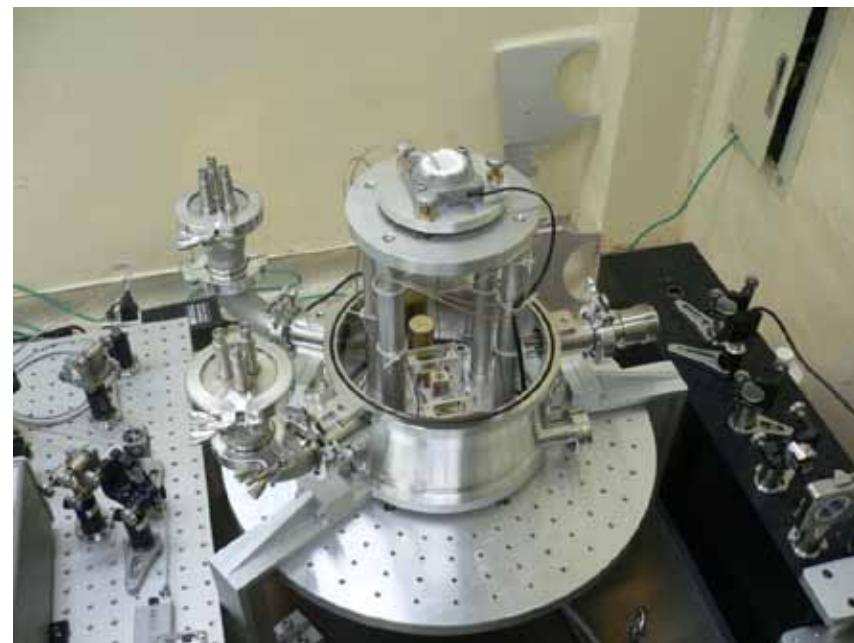
- One fundamental constant involved ~ gravitational constant G
- Masses of kg scale produce force at nanonewton level
- **Weakest in all the fundamental forces, and difficult**



$$F_g = GMm/r^2$$

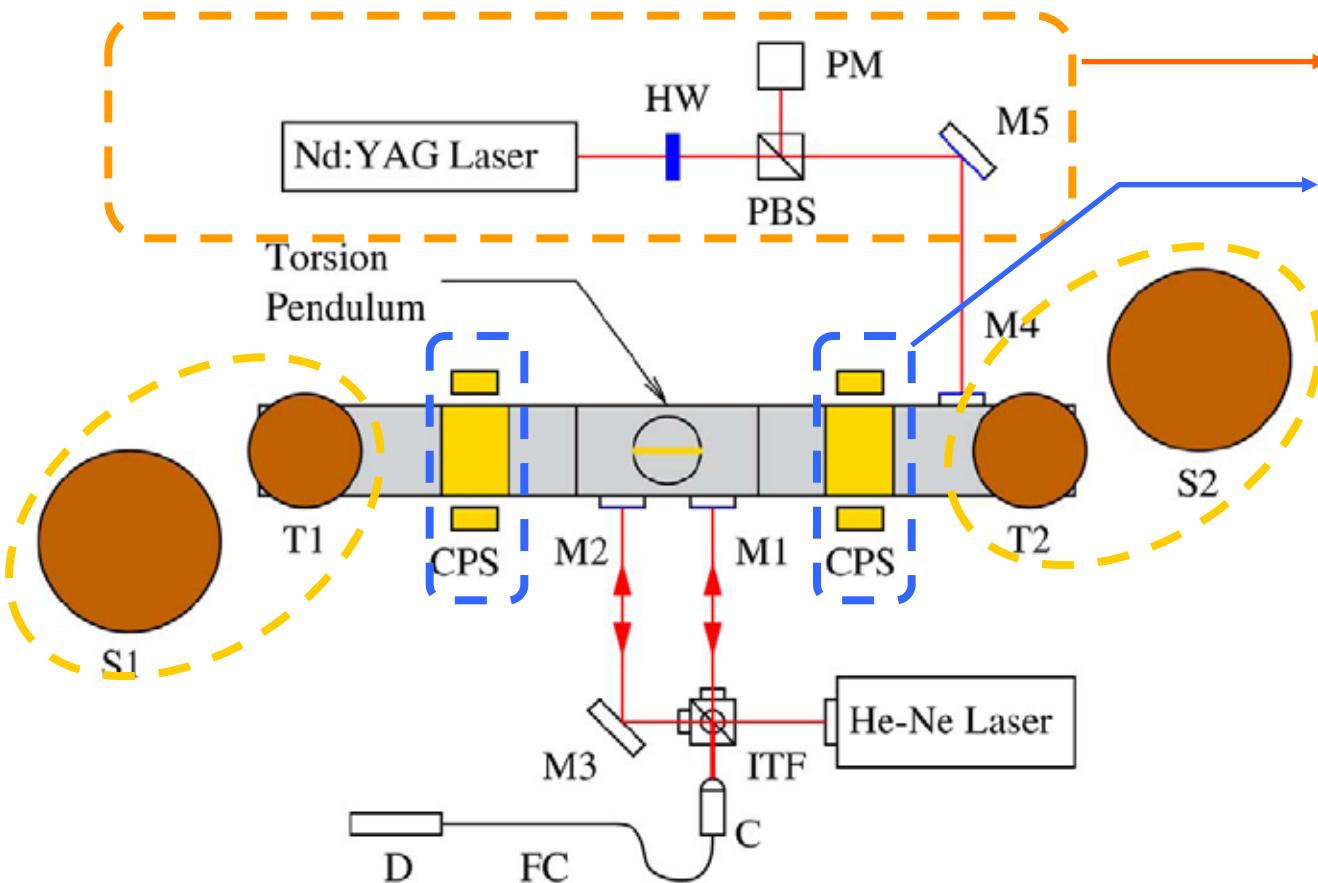
3-1. Torsion pendulum facility

- Force range: **horizontal force below 50 nN**
- Target resolution: **few piconewton**
- Mechanical spring: **strip torsion pendulum**
- Compensation force: **radiation pressure force, electrostatic force** and **universal gravitation**



Torsion pendulum facility

Top view:

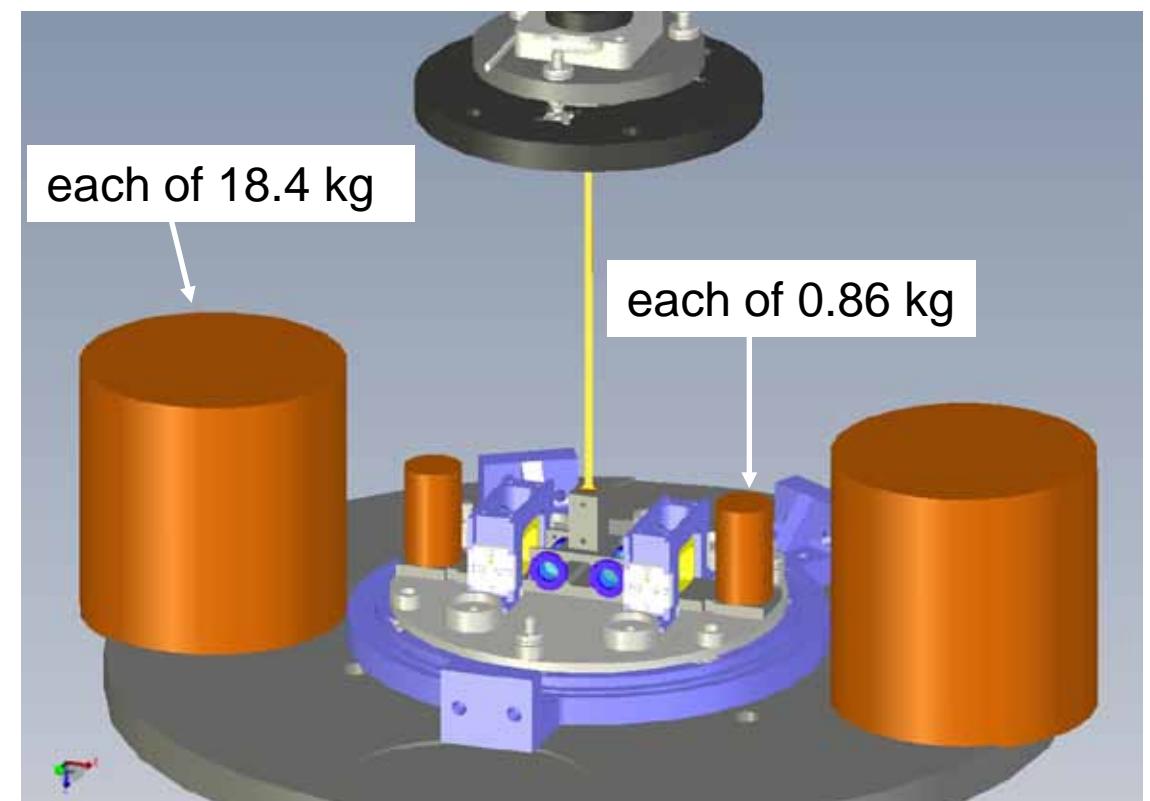


Radiation pressure force

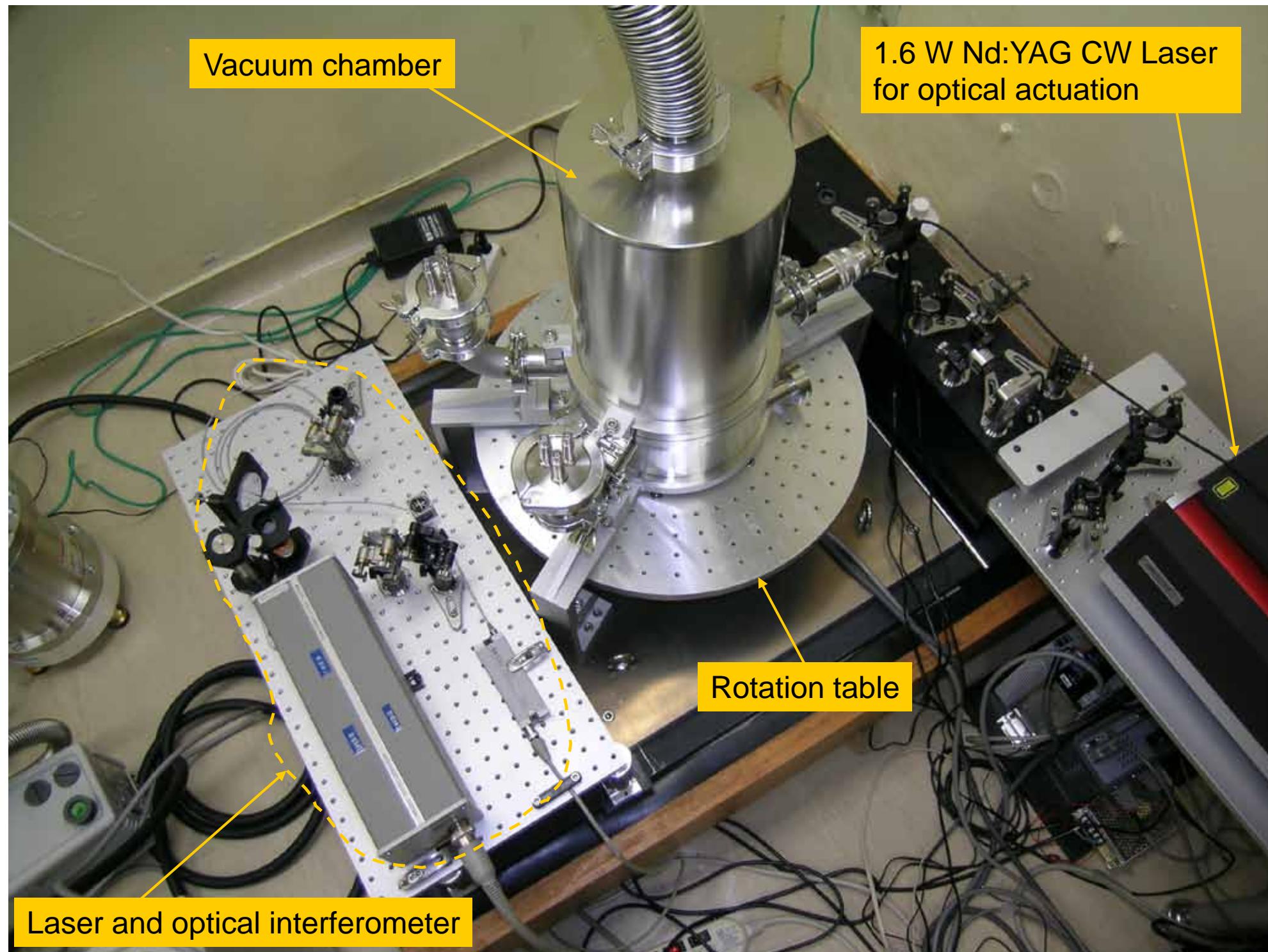
Electrostatic force

Universal gravitation

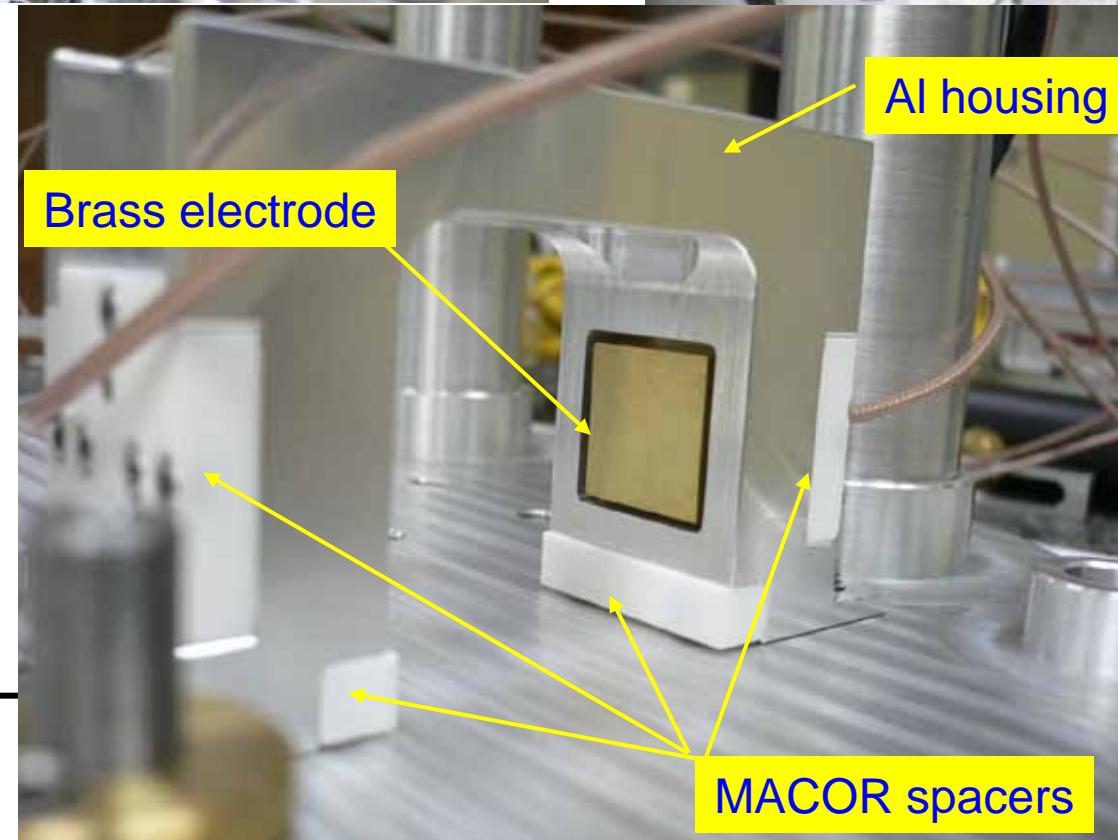
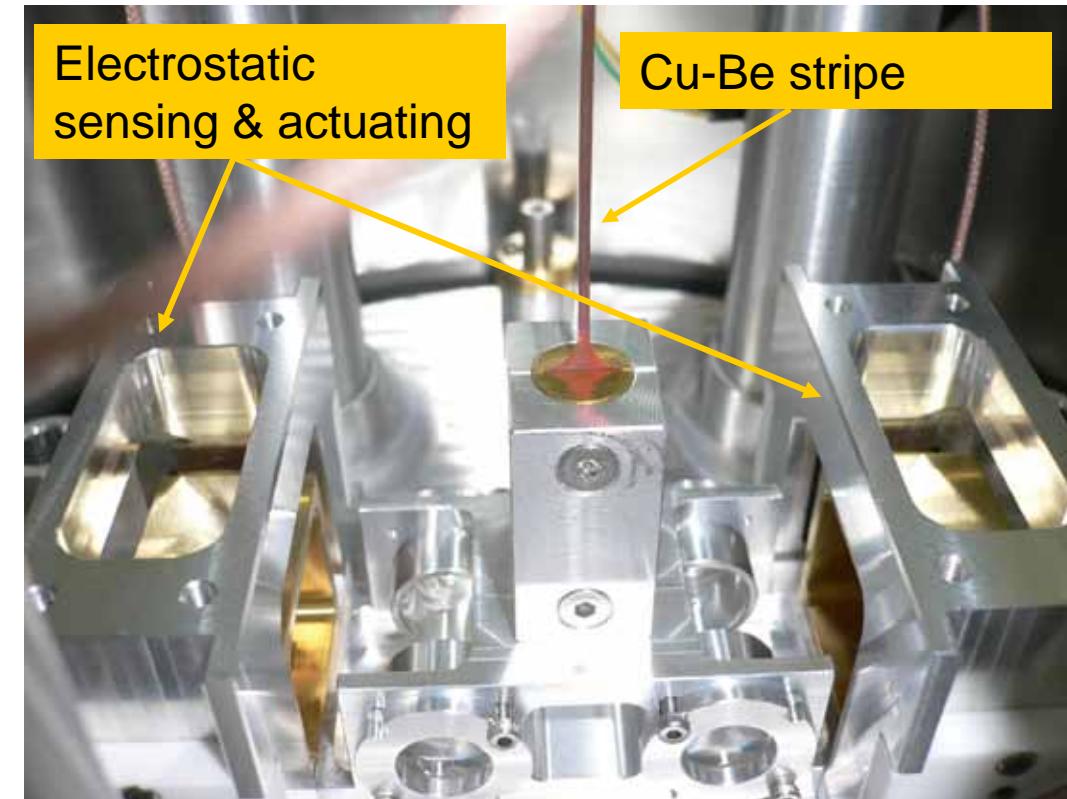
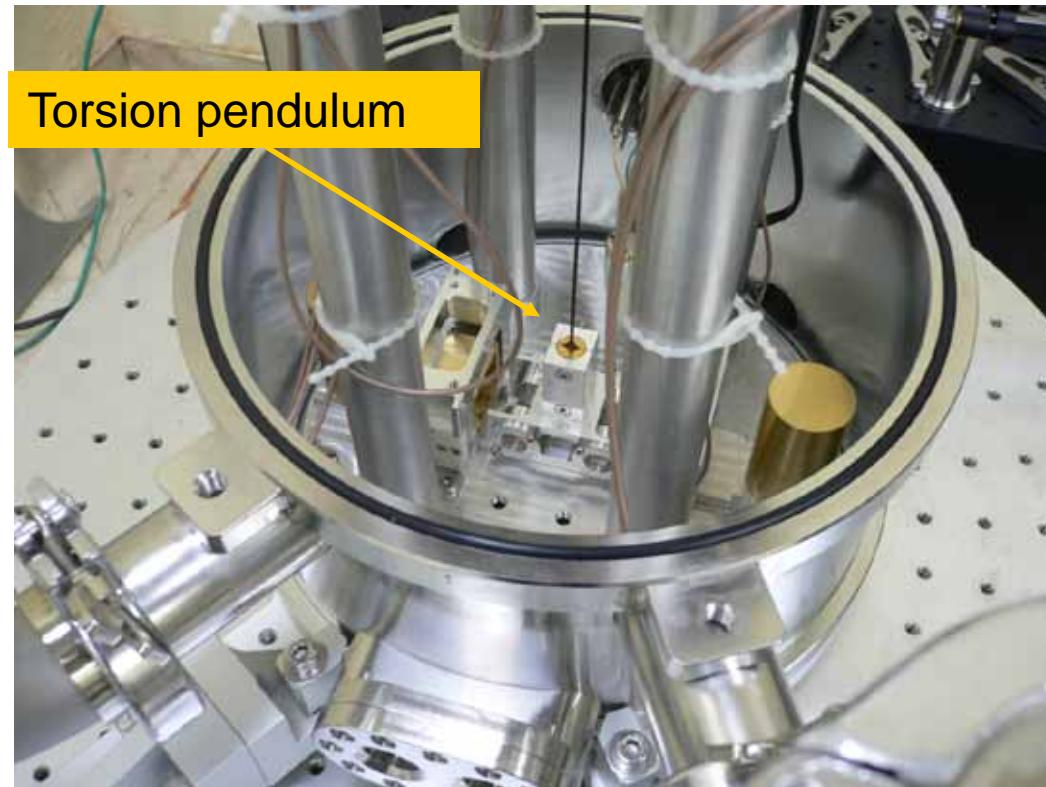
Schematic view of experimental setup. T1 and T2: Test masses. S1 and S2: Source masses. ITF: Interferometer. CPS: Capacitive position sensor. M1 to M5: Reflection mirrors. C: Collimator. FC: Fiber cable. D: Photodetector. PBS: Polarizing beam splitter. PM: Power meter.



Torsion pendulum experiment

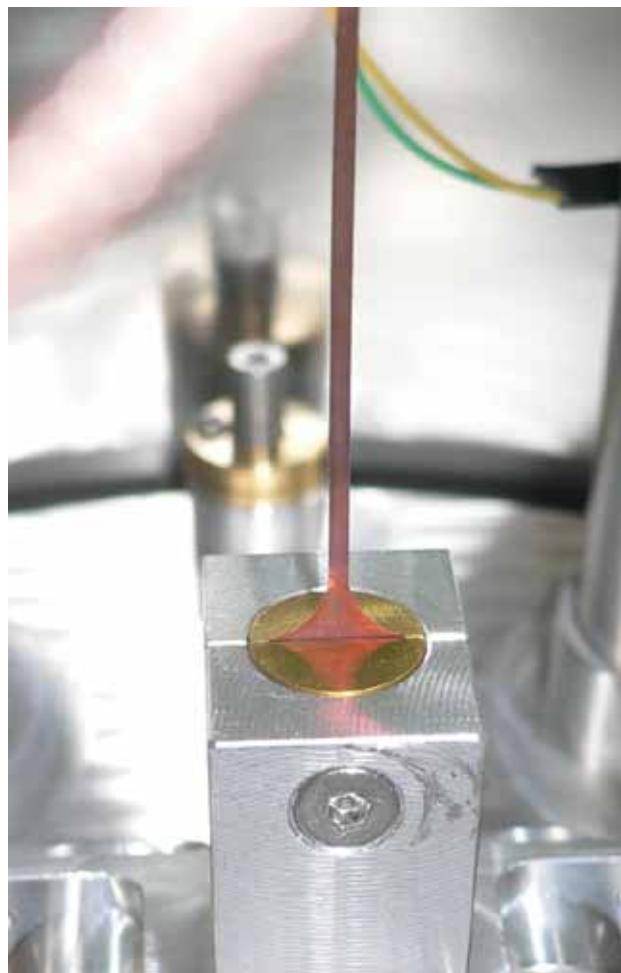


Torsion pendulum experiment



Strip torsion pendulum

Cu-Be strip



Thickness t : 27 μm

Width b : 1.2 mm

Length L : 200 mm

Spring constant is dominated by the **load**, not the **material's properties**

$$k_t = Mgb^2 / 12L + bt^3F / 3L$$

T.J. Quinn *et al*, *Metrologia*, 1997, **34**, 245-249

=> Reducing influence of noises resulting from thermal fluctuations and anelasticity

$$k(1 + if(\omega))$$

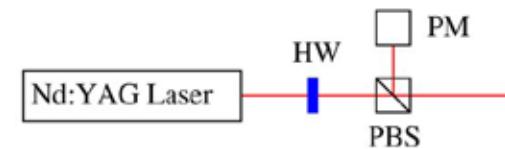
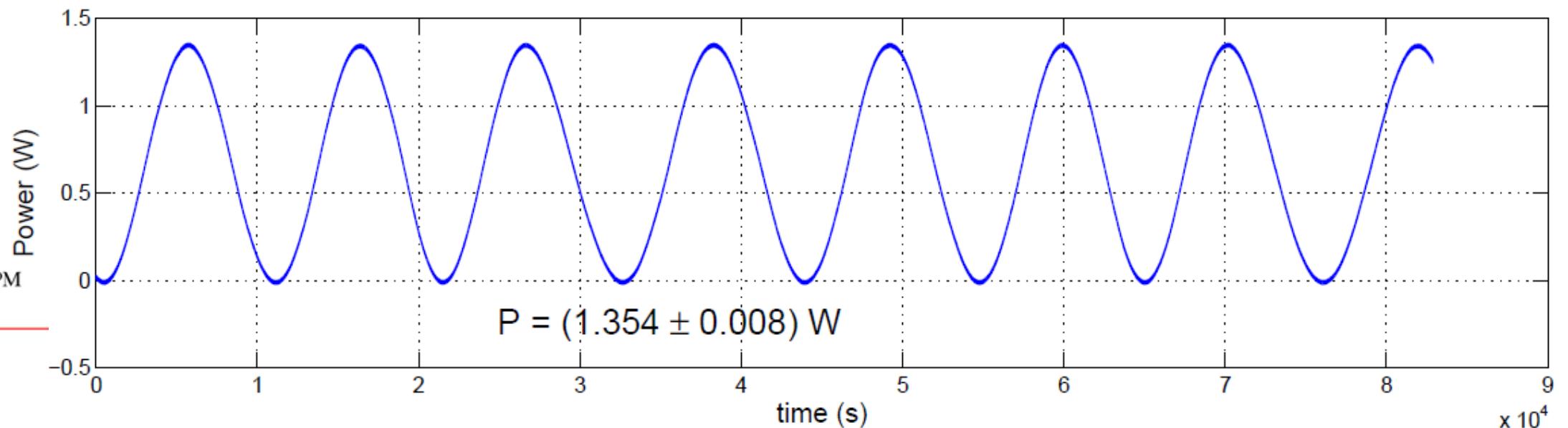
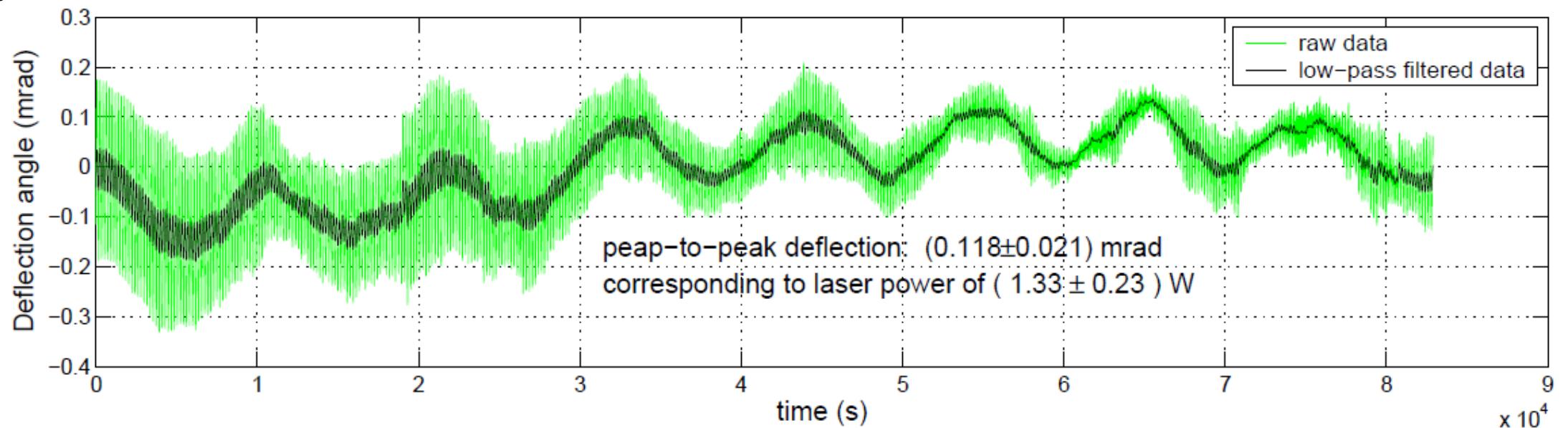
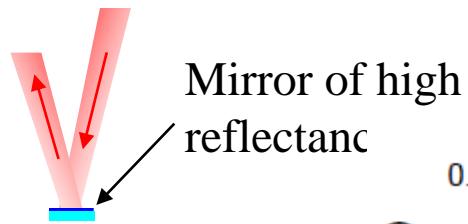
Torsional spring constant:

$$k_t = 5.25 \cdot 10^{-6} \text{ Nm/rad}$$

$$\text{Or } k_{\text{linear}} = k_t / l^2 = 0.001 \text{ N/m} \\ (\text{for } l = 70 \text{ mm})$$

F: shear modulus

Radiation pressure force



Damp the resonant motion without increasing the thermal noise $\mu \sqrt{4k_B T b}$
 => active damping

Active damping control using light pressure force

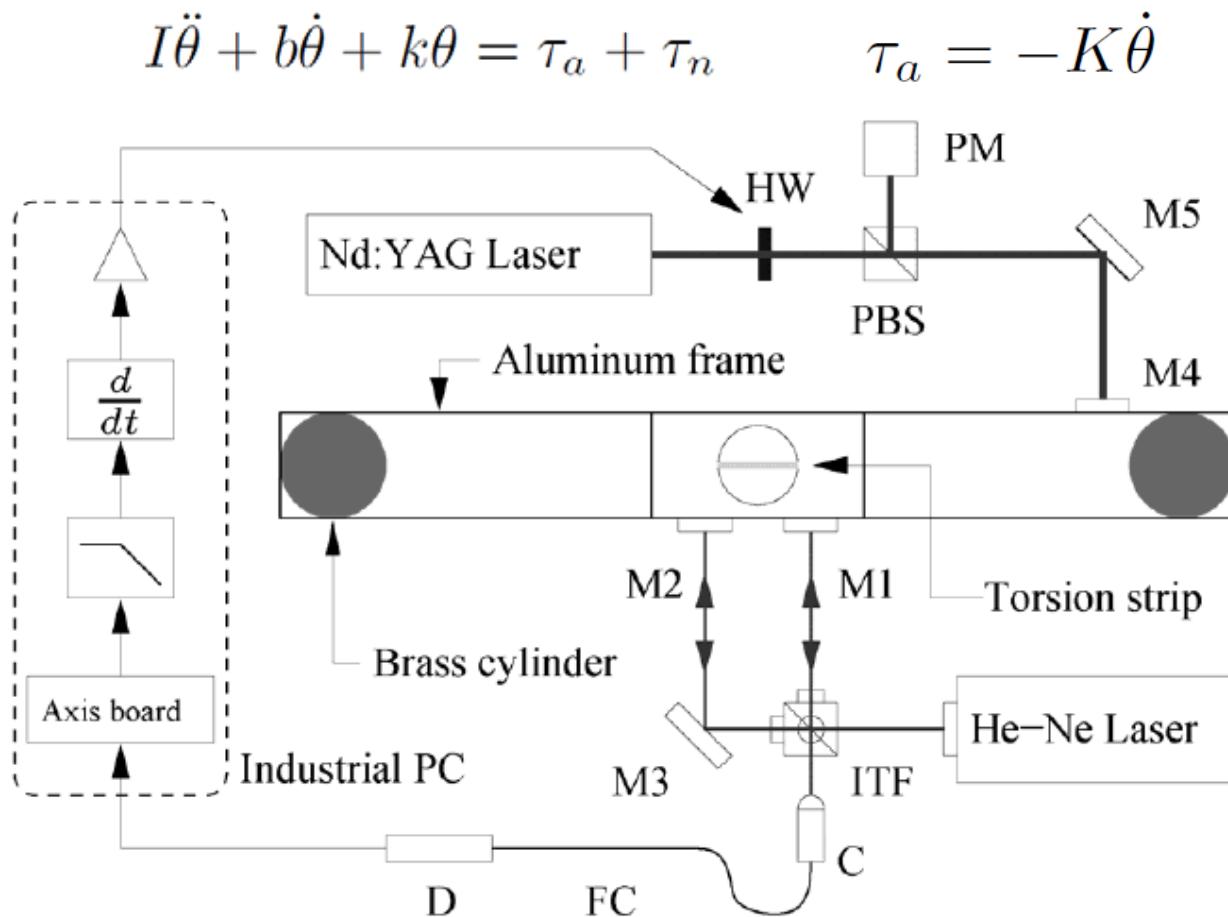


FIG. 1: Schematic view of experimental setup. ITF: interferometer; M1 to M5: reflection mirrors; C: collimator; FC: Fiber cable; D: photo-detector; PBS: polarizing beam splitter; PM: power meter.

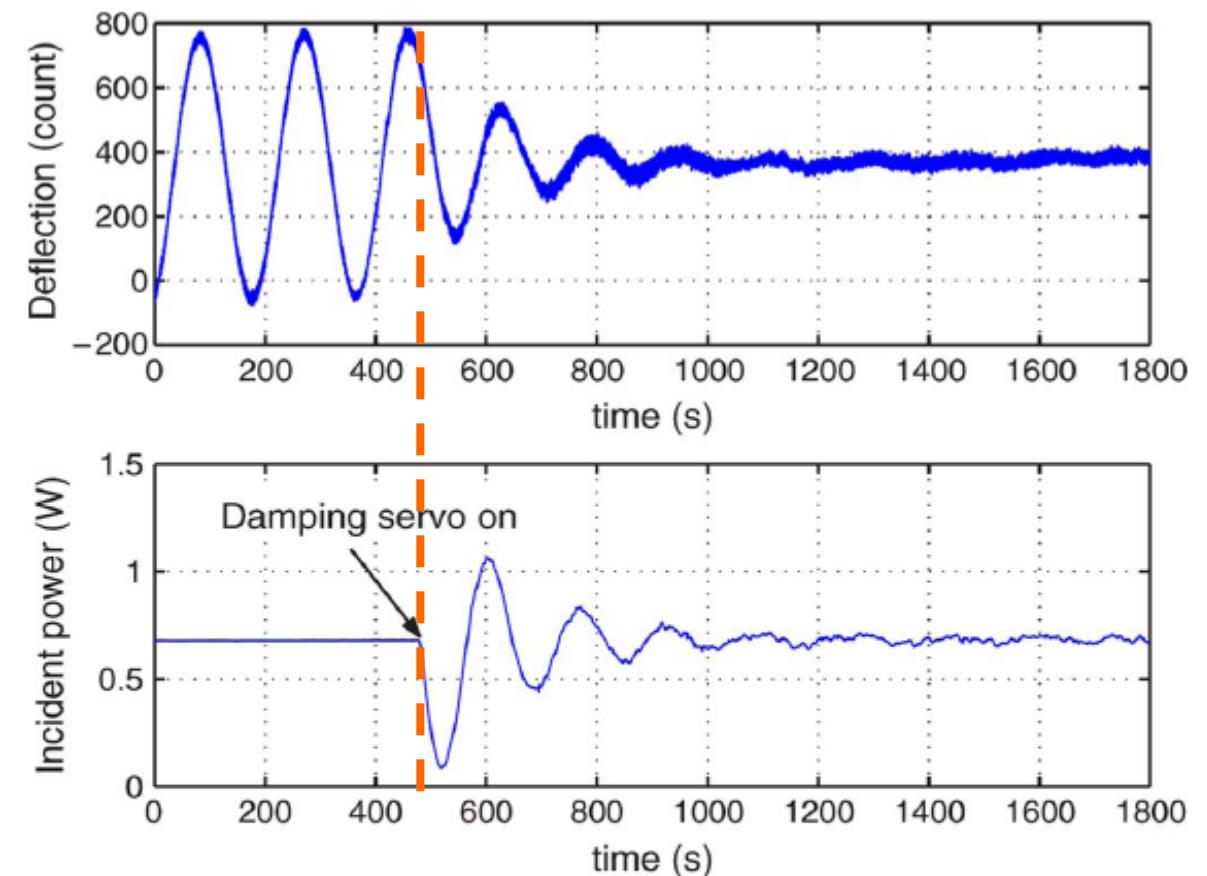
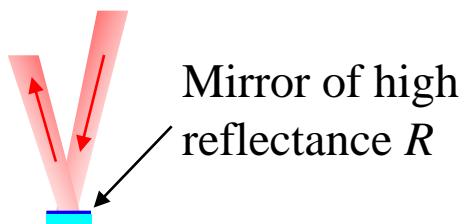
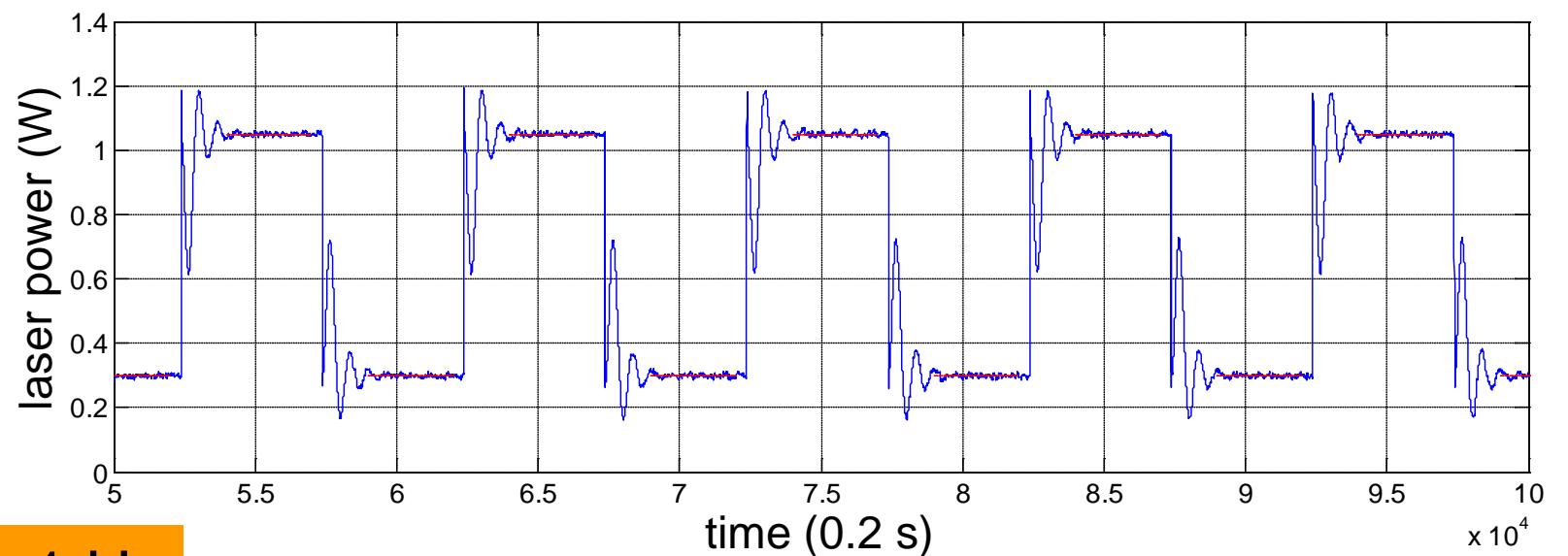


Fig. 5. Damping created by the radiation pressure force. (Top) Deflection measured by the interferometer (one count corresponds to 10 nm). (Bottom) Incident power on the mirror M4. After the damping servo was switched on, the incident power was varied to generate the torque needed to damp the pendulum's motion.



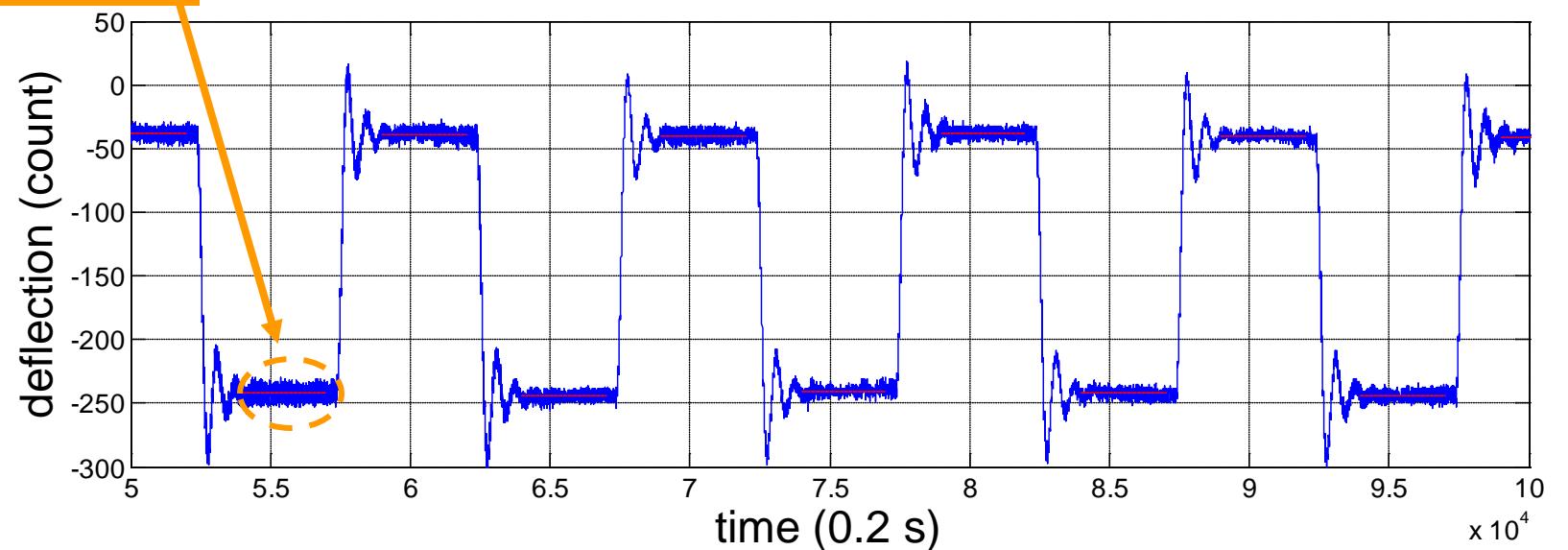
Radiation pressure force

$P=0.75\text{ W}$
 $2P/c=5.0\text{ nN}$



Small pendulum motion at $\sim 1\text{ Hz}$

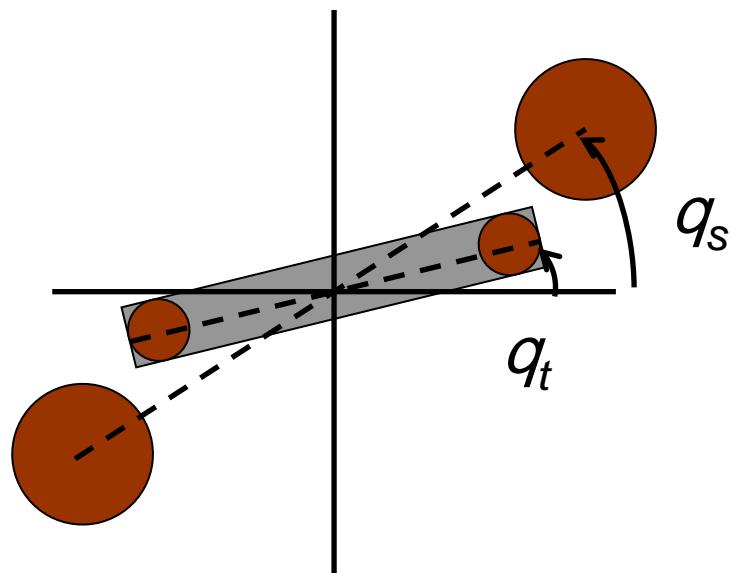
Measurement:
 $4.9 \pm 0.4\text{ nN}$



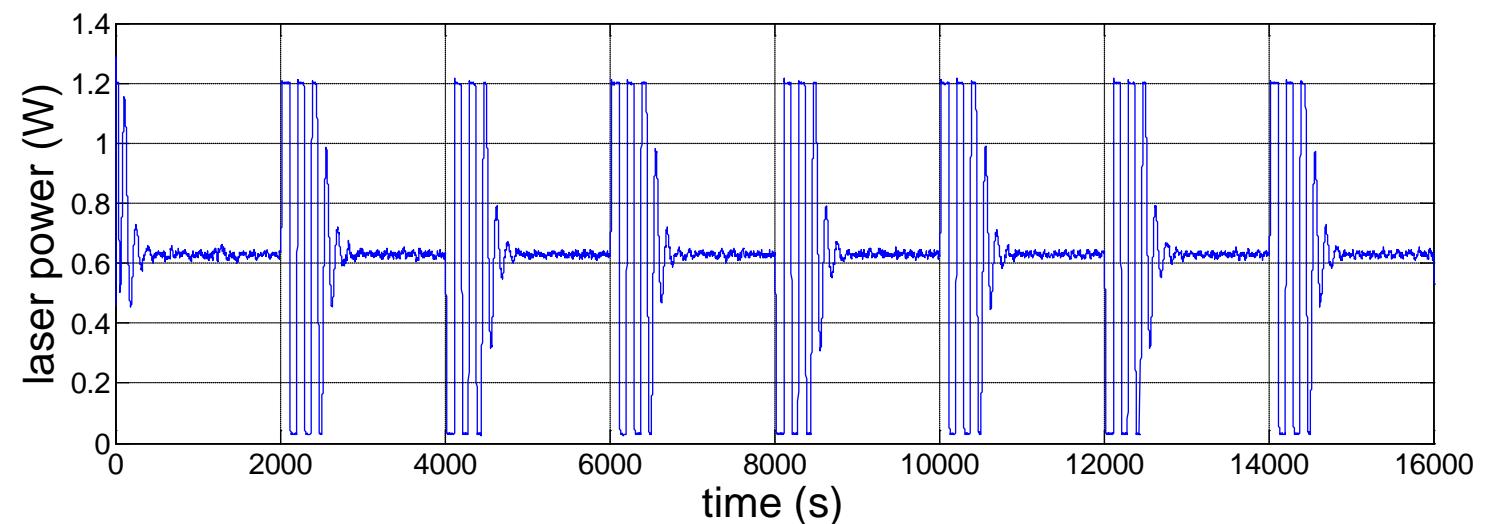
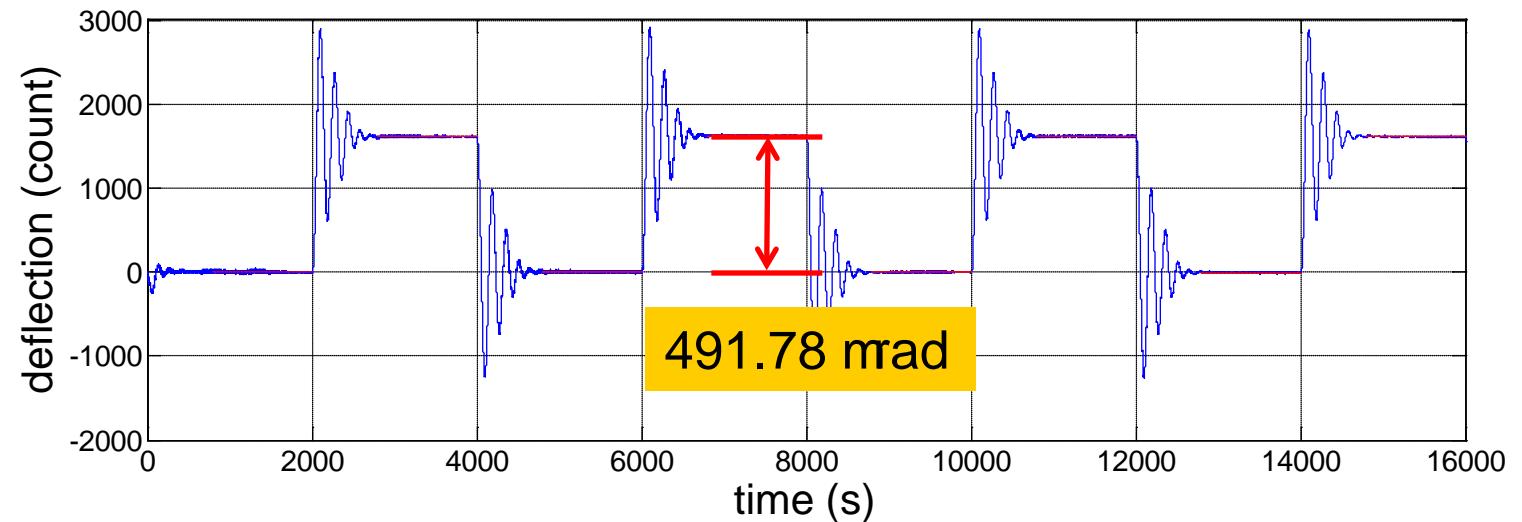
S-J Chen and S-S Pan, "Nanonewton force generation and detection based on a sensitive torsion pendulum", IEEE Trans. Instru. Meas. **58** (2009) 897-901

Optical pressure Calibration by Universal Gravitation

To have the torsion pendulum quickly reach its equilibrium position, radiation pressure force was used to damp the resonant motion.



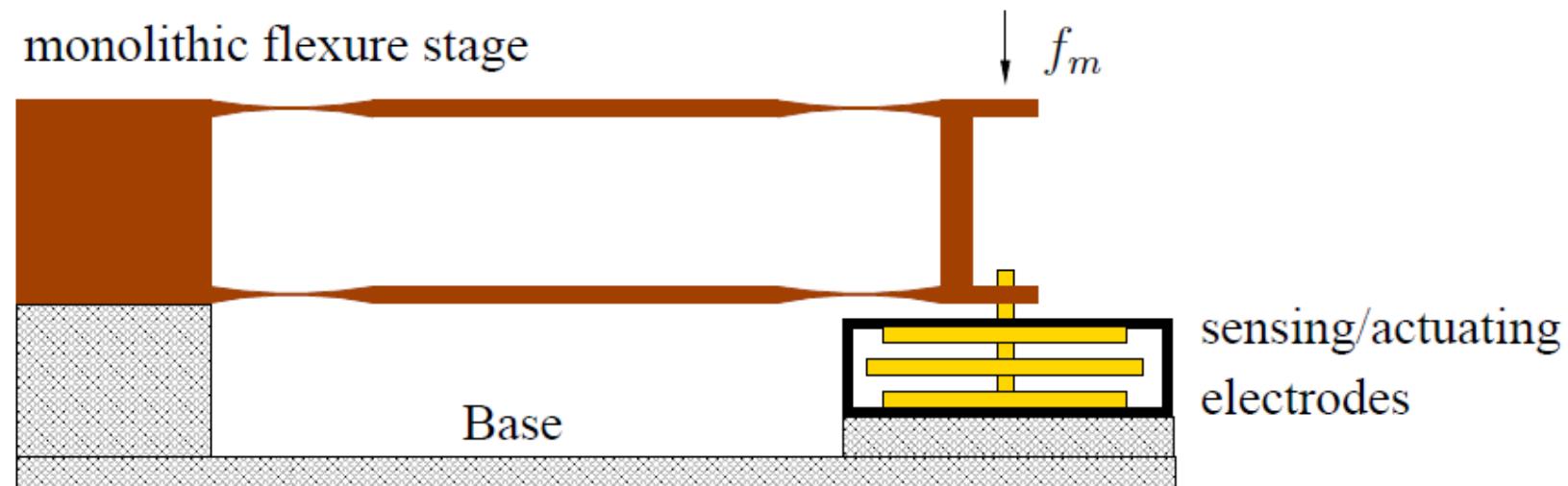
$$F_g = GMm/r^2$$



Work is in progress.....

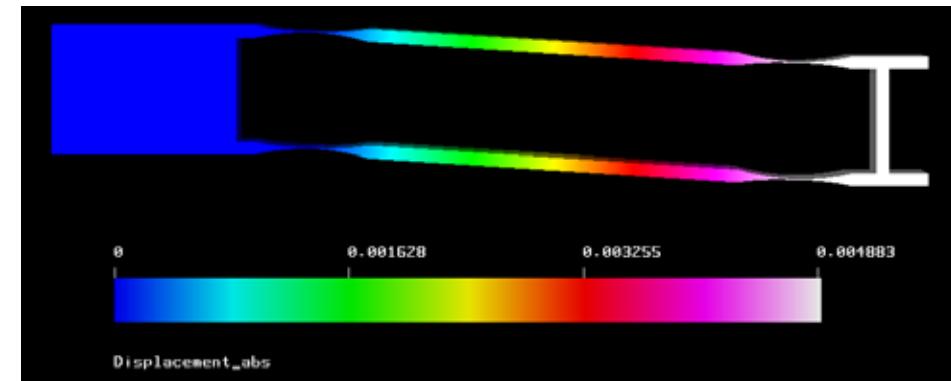
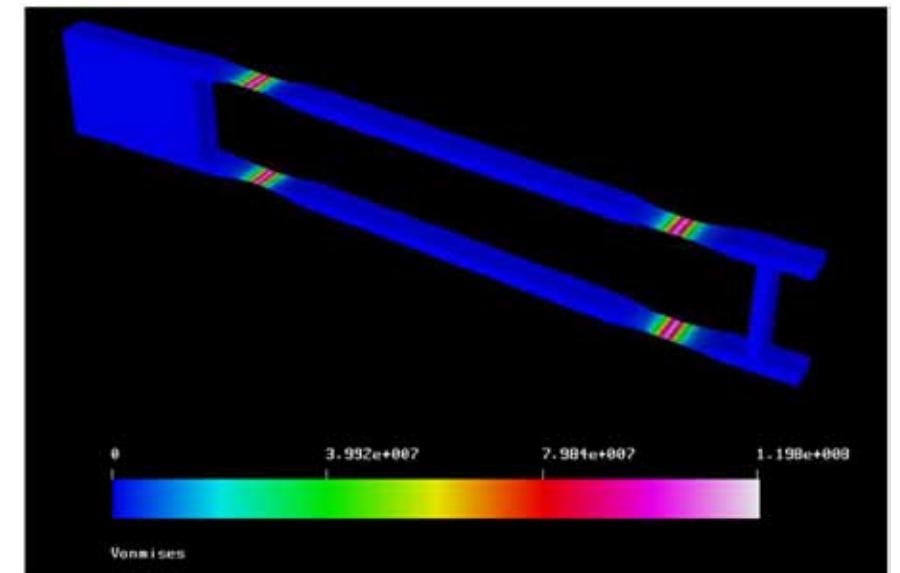
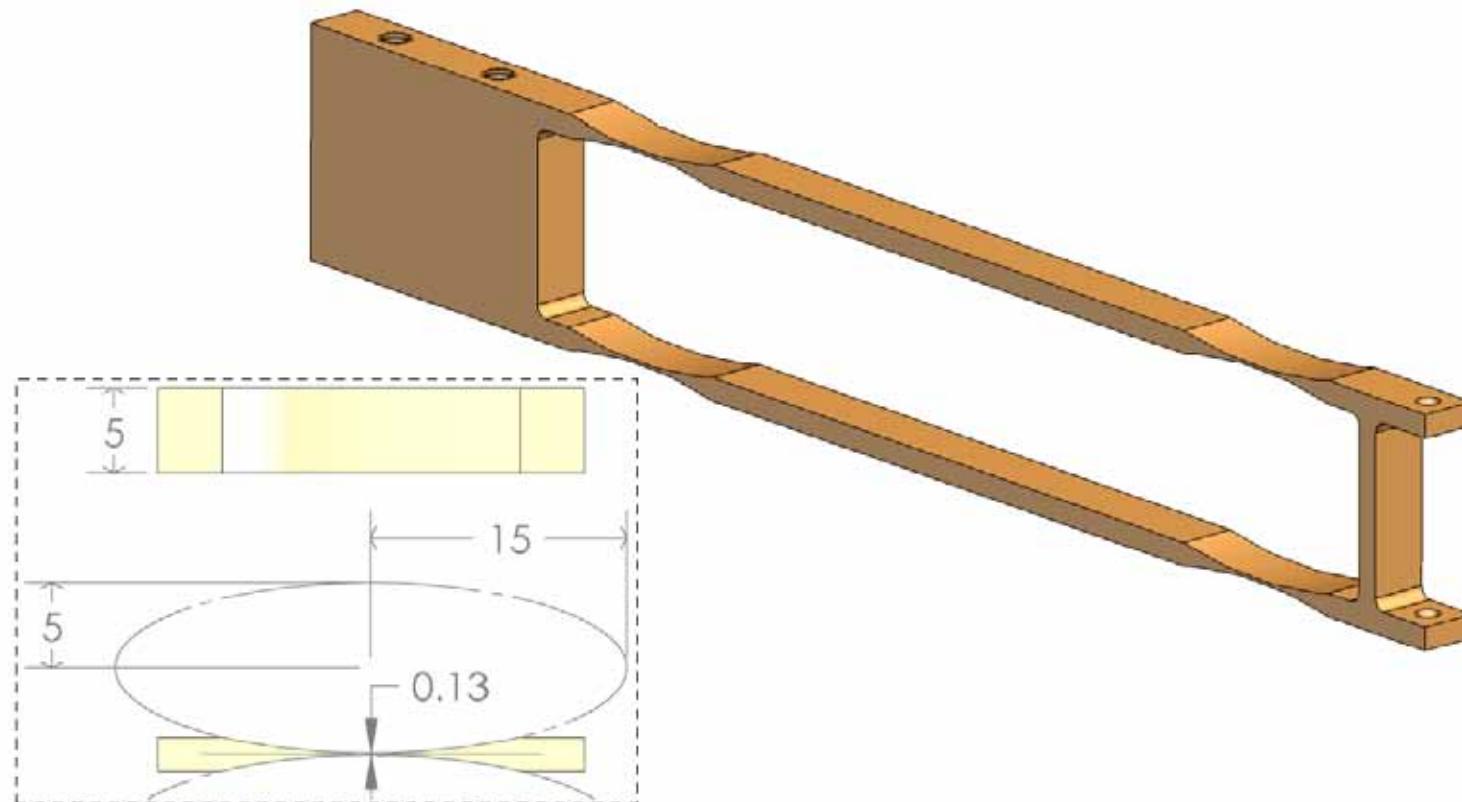
3-2. Flexure stage with electrostatic sensing and actuating

- Force range: vertical force below 200 mN
- Target resolution: few nanonewton
- Mechanical structure: monolithic flexure stage
- Compensation force: electrostatic force



Monolithic flexure structure

- Flexure-pivot: elliptical notch type
- Four pivots for rectilinear motion
- Material: aluminum alloy (Al 6061-T6), copper-beryllium (C-17200)
- Stiffness: 16.32 N/m and 19.1 N/m



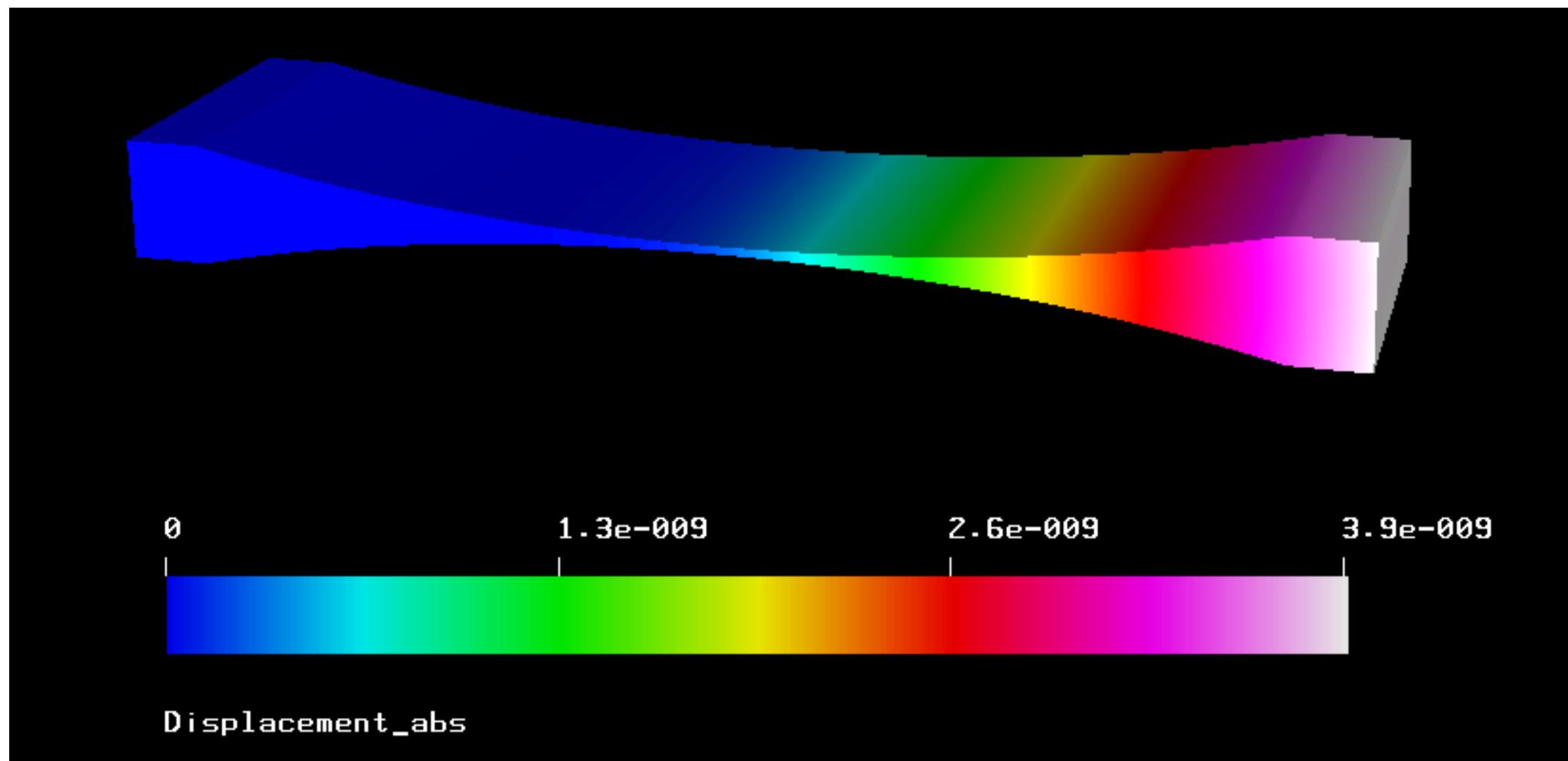
Low stiffness flexures

Elliptical notch flexure, thickness ~ 0.13 mm



Low stiffness flexure

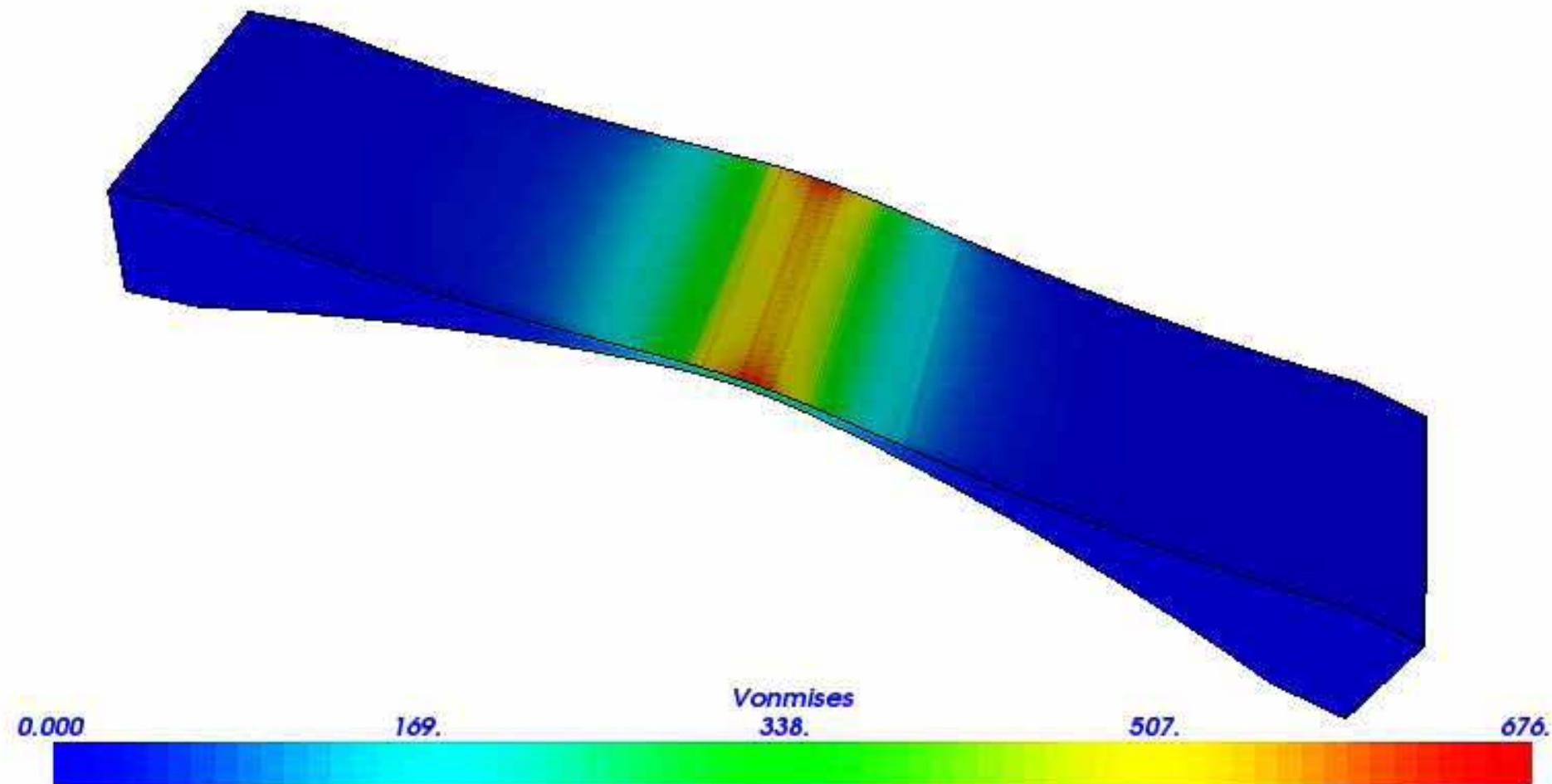
- Finite element analysis



The simulated displacement under a load of 1 mN. The displacement unit is in meter.

Low stiffness flexure

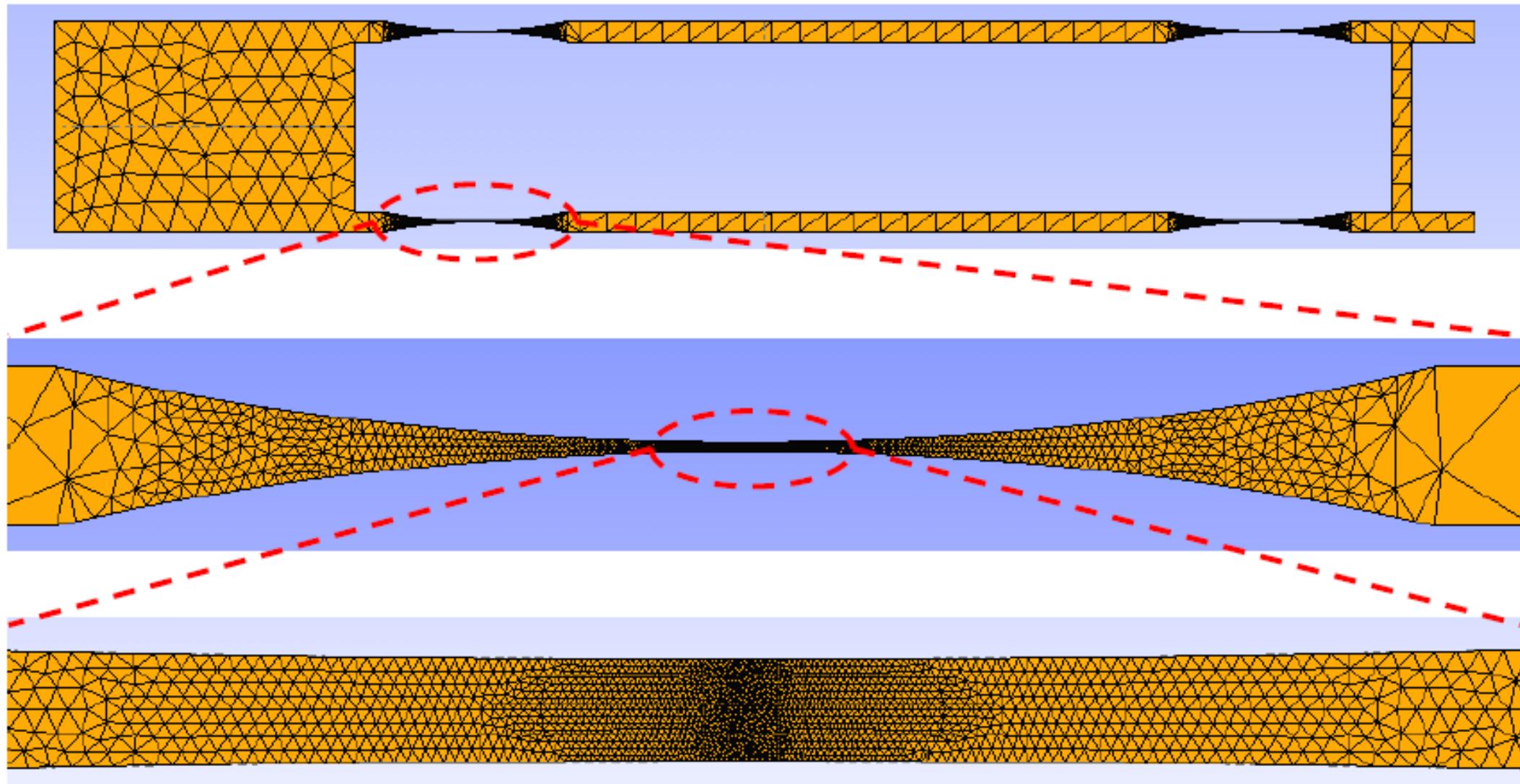
- Finite element analysis



Stress distribution under a load of 1 mN. The maximum stress is localized at the thinnest portion of the hinge. The stress unit is in N/m².

Low stiffness flexure

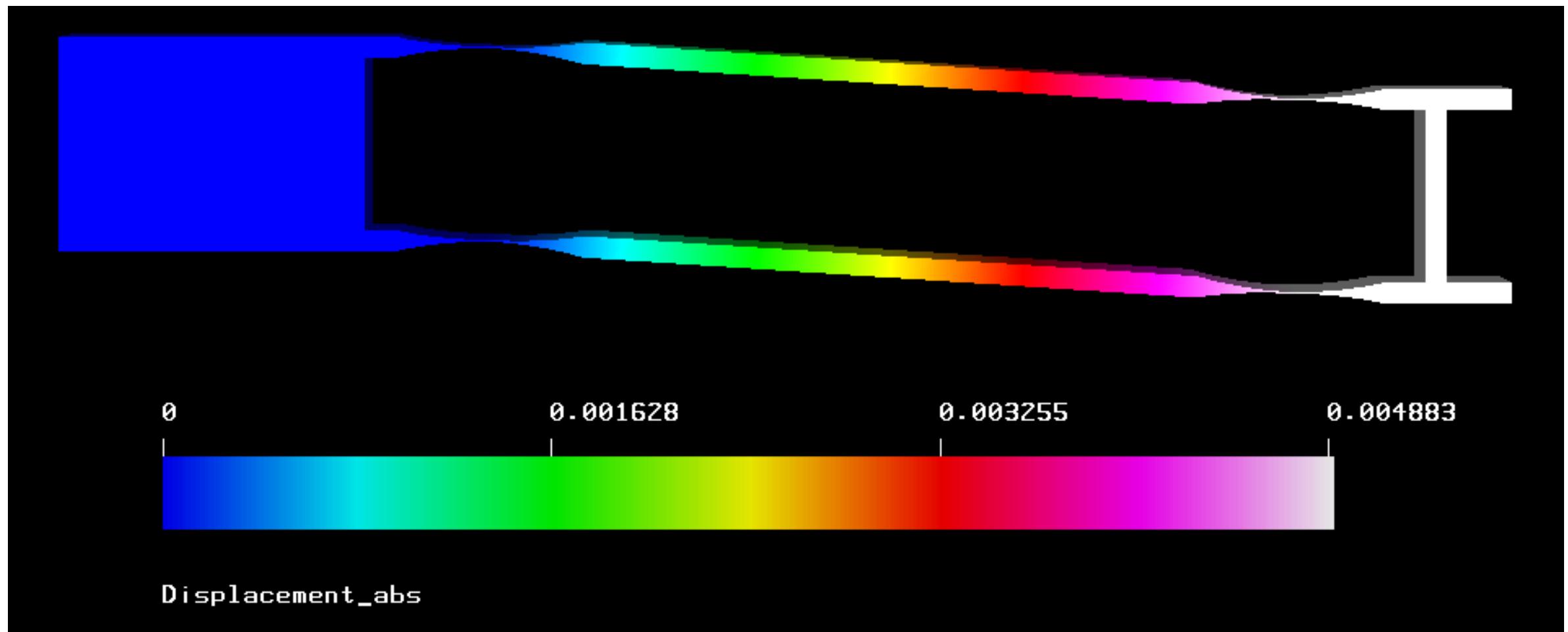
- Finite element analysis



The mesh model of the structure constructed by 20,467 volume elements with a element size ranging from 0.5 mm to 5 mm.

Low stiffness flexure

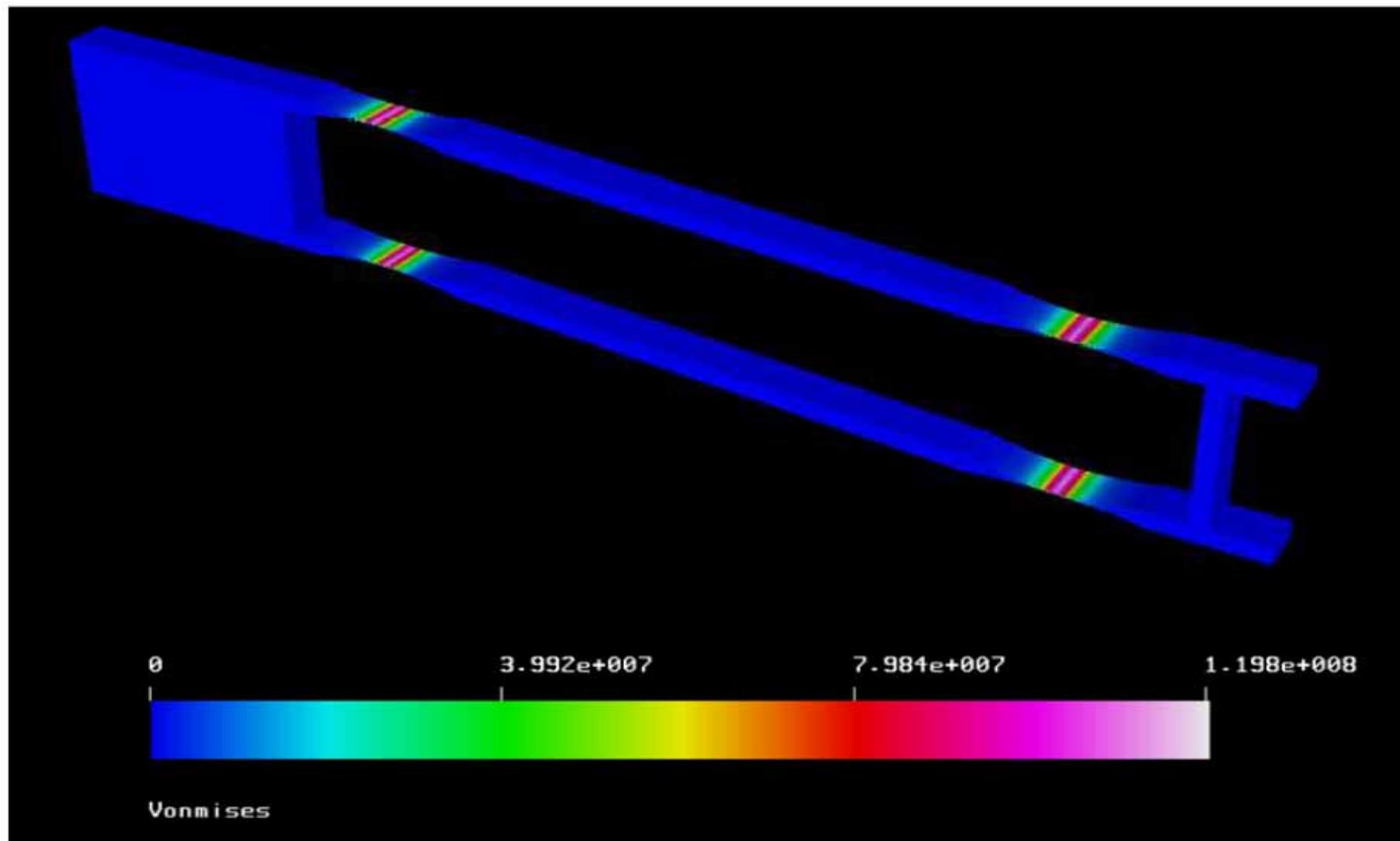
- Finite element analysis



The deflection due to the structure's bodyweight and the load of disk electrode (about 6.93 g).

Low stiffness flexure

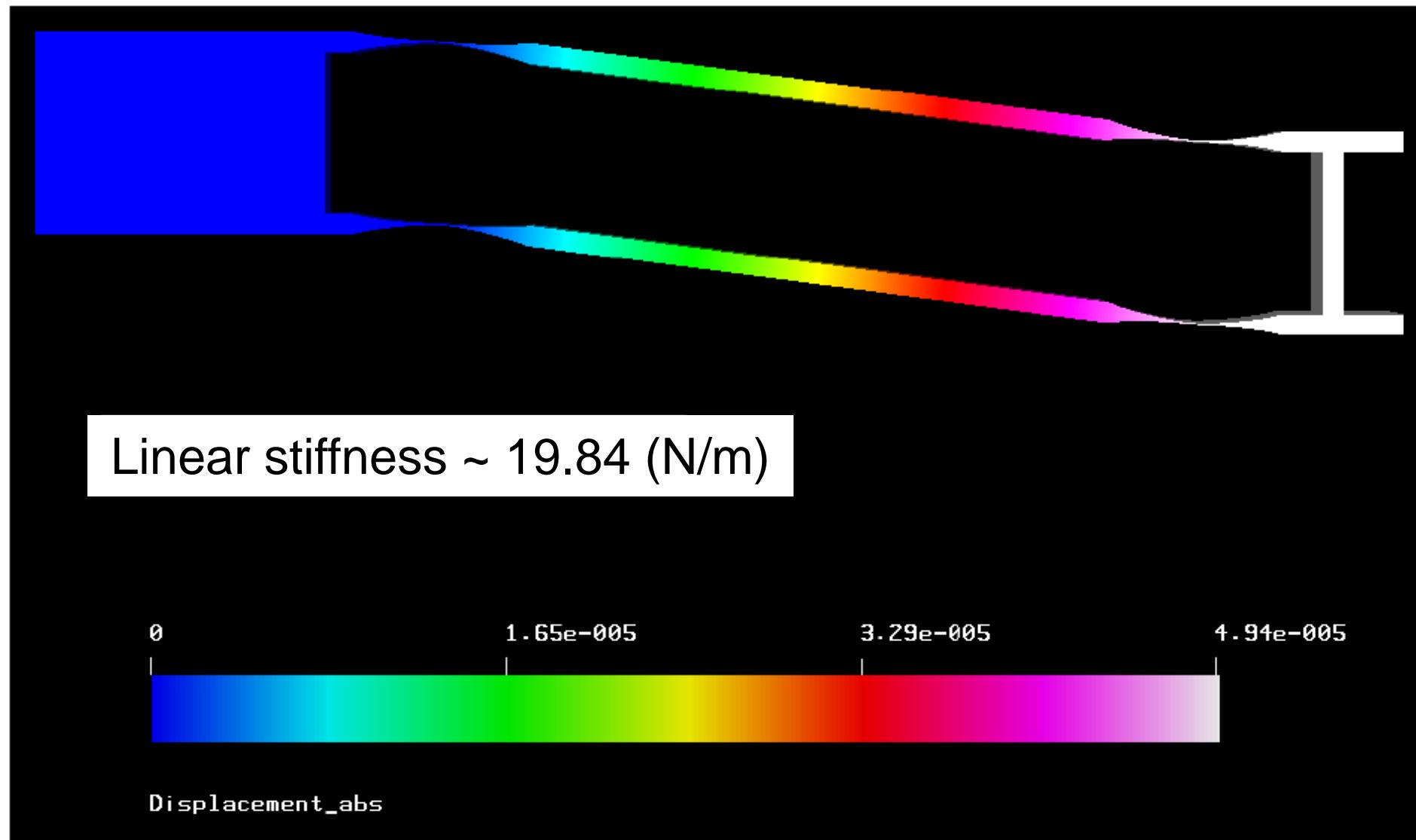
- Finite element analysis



The stress due to the structure's bodyweight and the load of disk electrode.

Low stiffness flexure

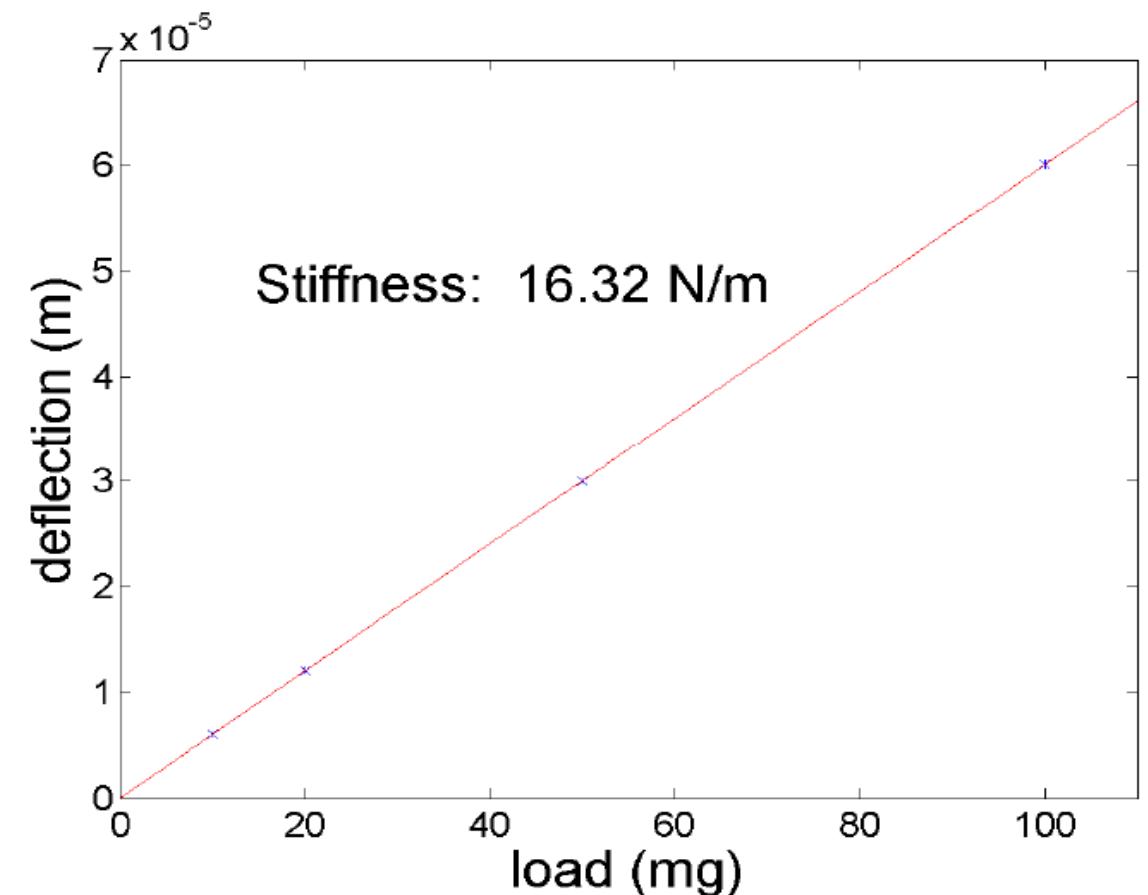
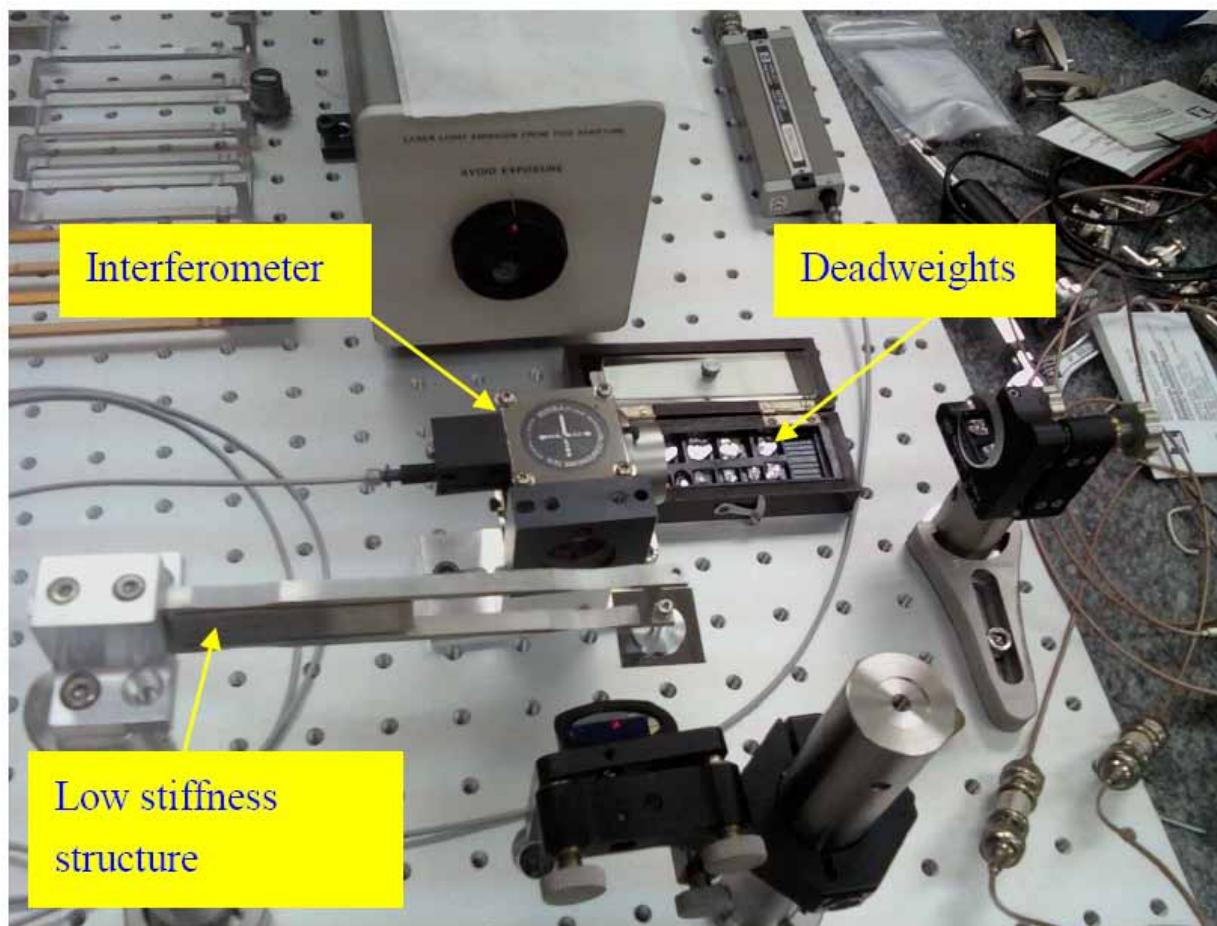
- Finite element analysis



The deflection under a load of 100 mg.

Low stiffness flexure measurement

- Stiffness measurement



Force balance control

Differential capacitance sensing:

$$DC = C_1 - C_2 \gg \frac{2\epsilon A}{d^2} x$$

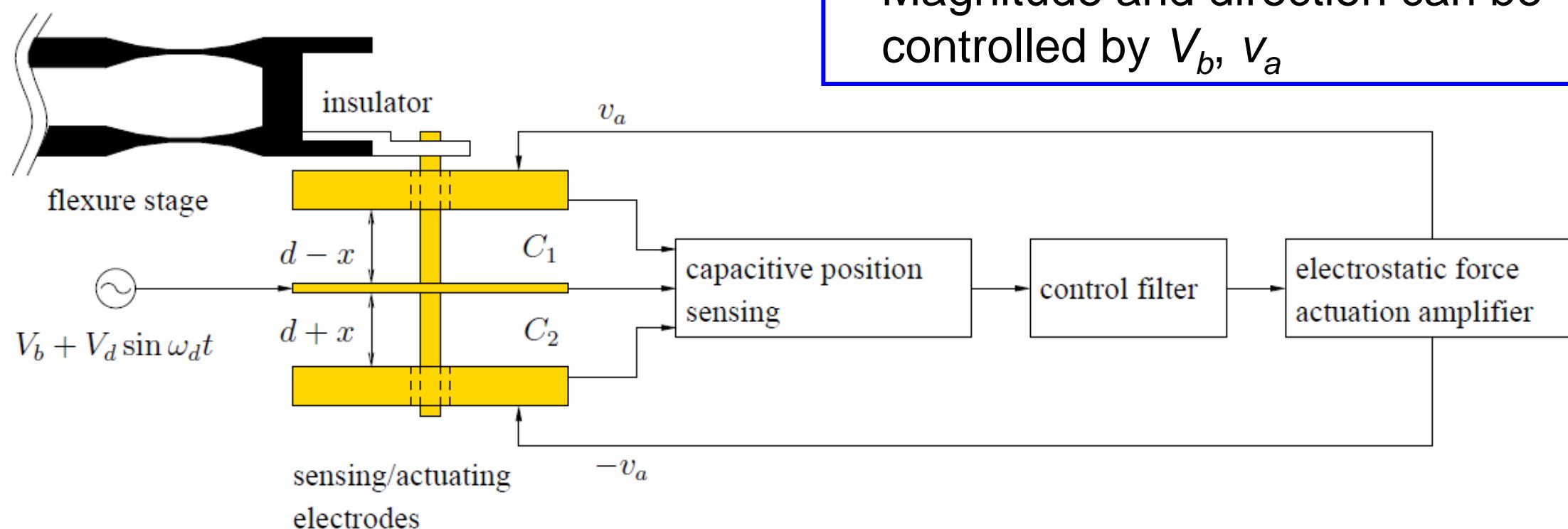
Measured at high frequency (100 kHz)

Electrostatic force actuation:

$$f_e = \frac{1}{2} \frac{dC_1}{dx} (V_b + v_a)^2 - \frac{1}{2} \frac{dC_2}{dx} (V_b - v_a)^2$$

$$= 2 \frac{dC_1}{dx} \Big|_{x=0} V_b v_a$$

Magnitude and direction can be controlled by V_b , v_a

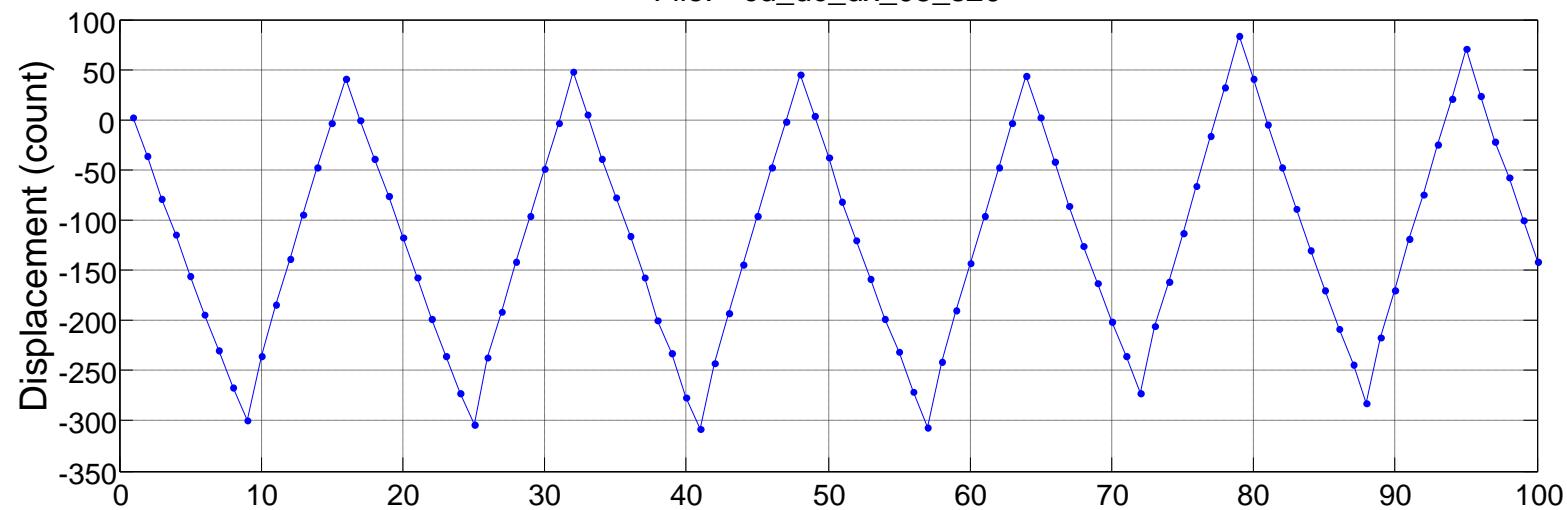


Deflection is maintained at zero by a FPGA based realtime controller (loop rate 200 kHz)

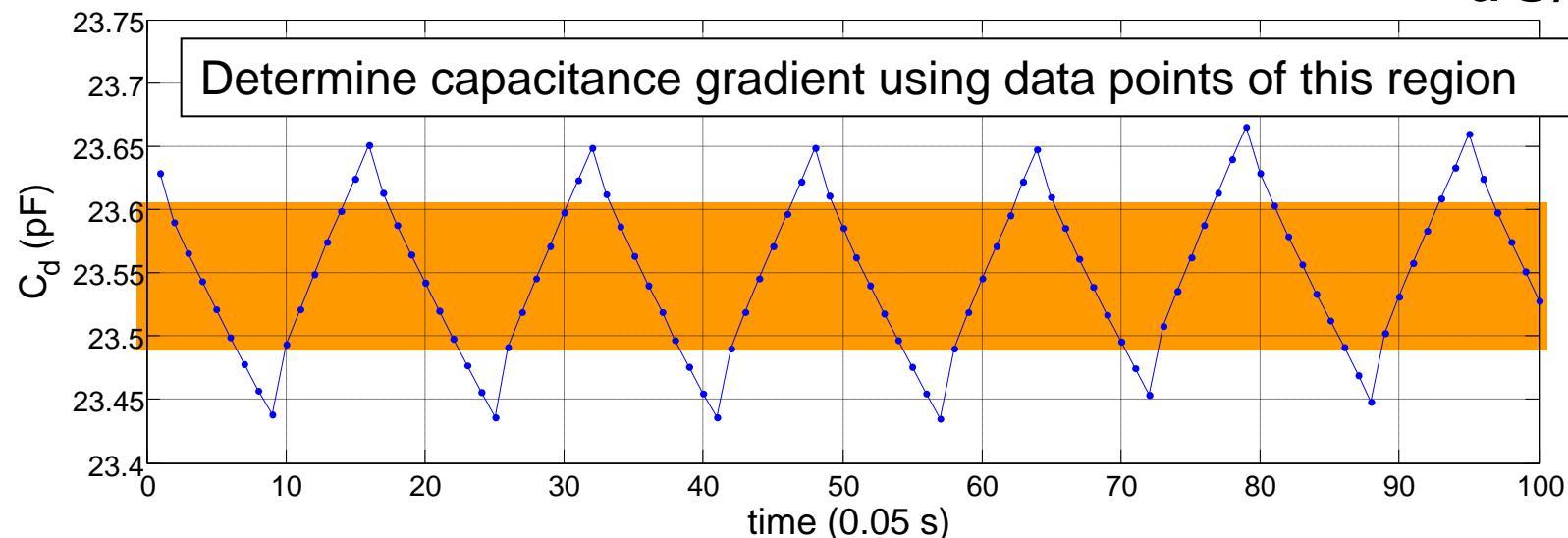
Capacitance gradient dC/dx

- Traceable measurement: calibrated capacitance bridge and optical interferometer.

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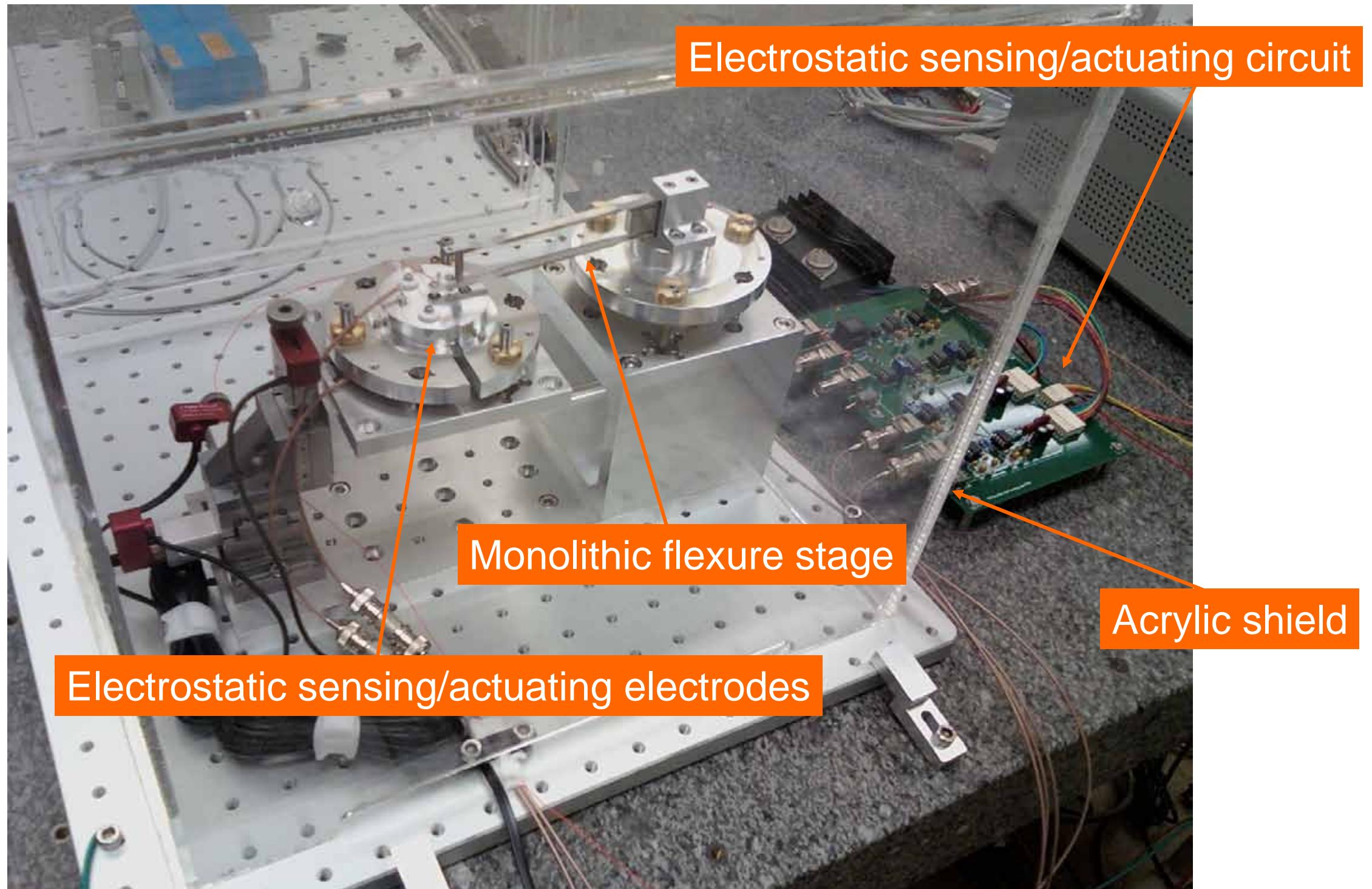


$$\Rightarrow dC/dx = (5.46 \pm 0.13) \cdot 10^{-8} \text{ F/m}$$



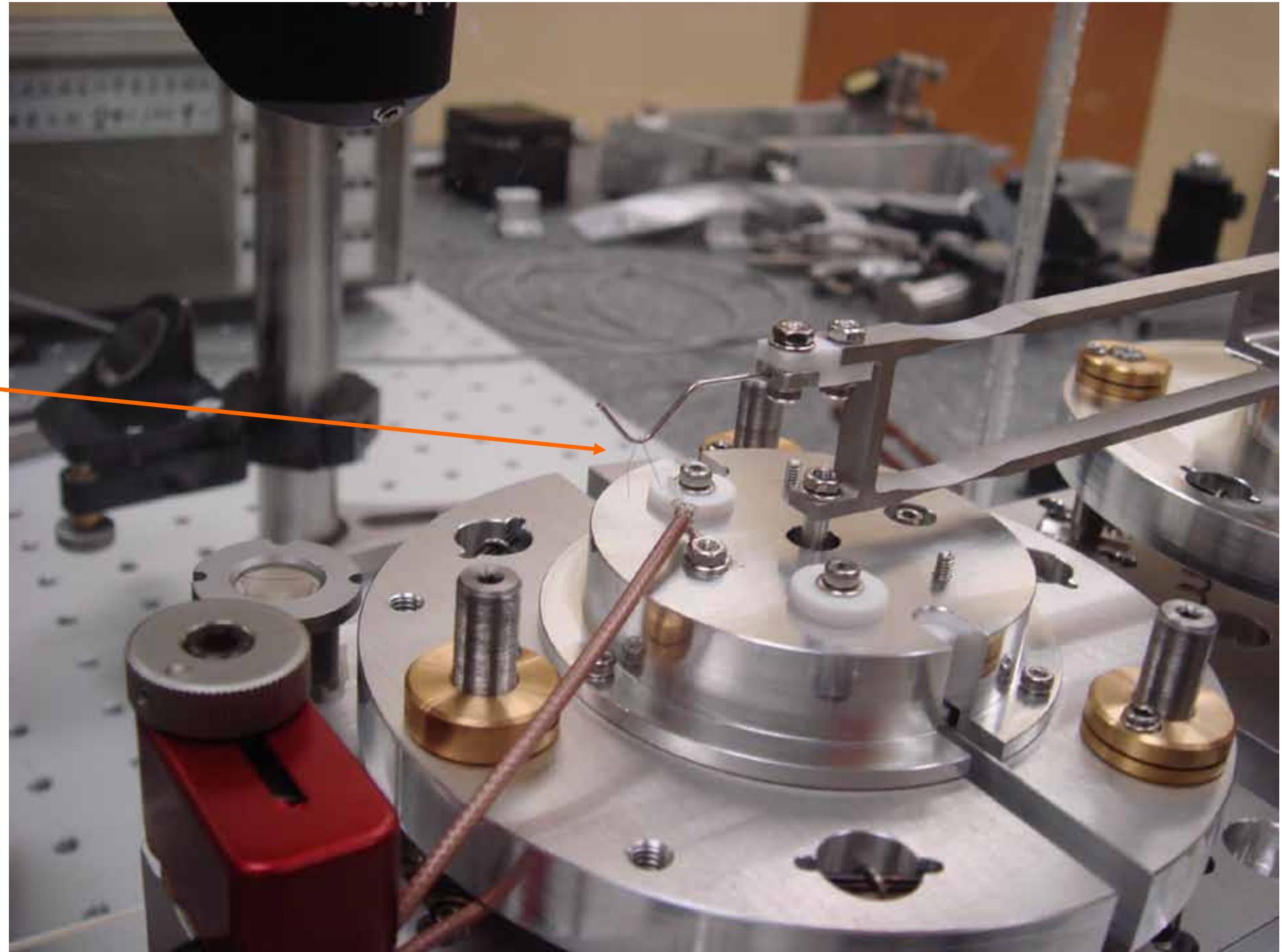
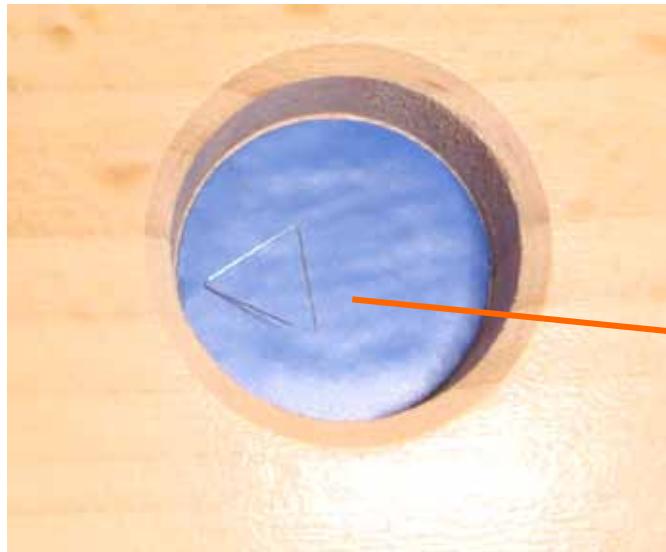
Average slope determined by the linear fit of more than 5000 capacitance to displacement data sets.

Force balance structure



1 mg weighing test (in air)

OIML class E1,
1 mg wire weight

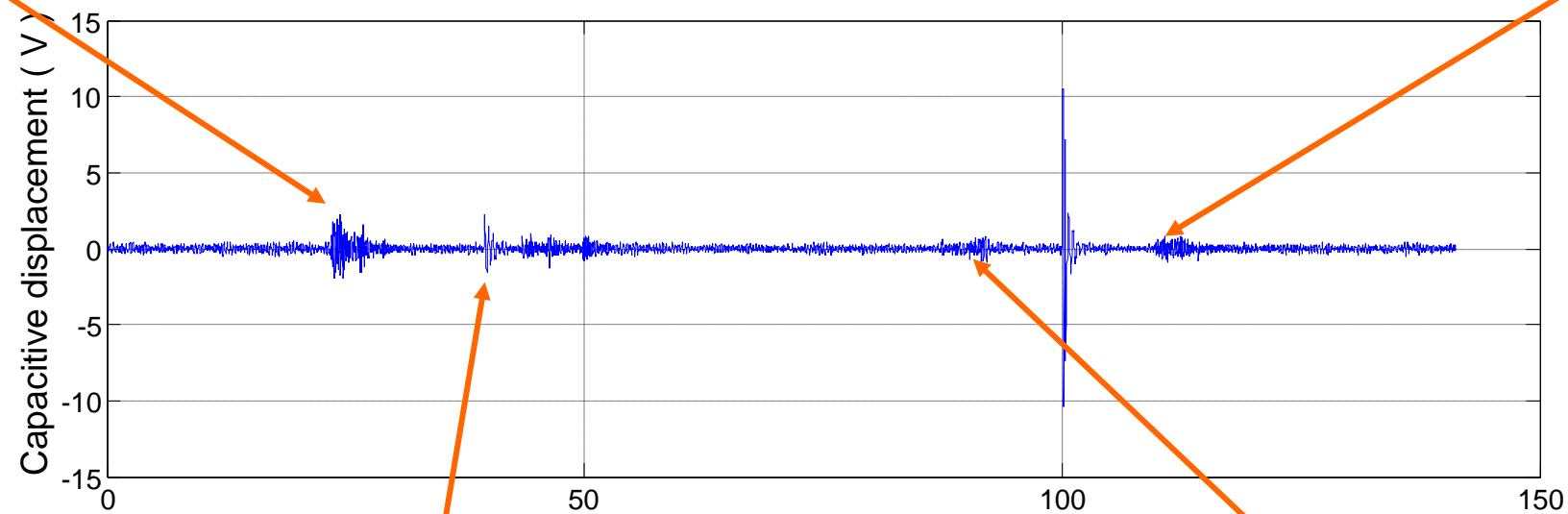


1 mg weighing test result

- For the moment, mass is loaded by hand

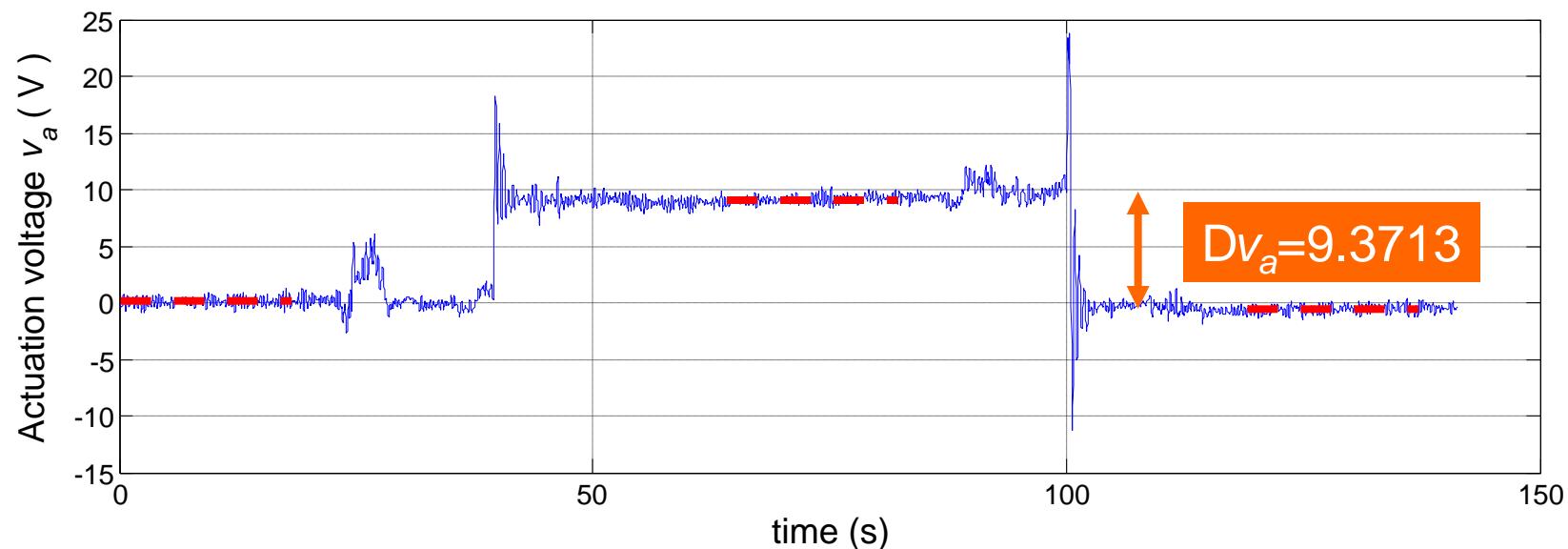
Open acrylic shield

Close acrylic shield



Load the weight & close the shield

Open the shield & unload the weight



1 mg weighing result

- $f_e = 2 V_b v_a dC_1/dx|_{x=0} = m_e g$

Measurement No.	f_e (mN)	Mass difference (mg)
1	9.7219	-6.9
2	9.802	1.3
3	9.8568	6.9
4	9.7089	-8.2
5	9.8504	6.3

Several tens of nanonewton

Mass difference is below 1%, test was performed in air and mass was loaded and unloaded by hand.

4. Summary

- Torsion pendulum facility and flexure stage with electrostatic force compensation introduced
- Radiation pressure force demonstrated, force sensitivity verified
- 1 mg weighing test in air completed, difference ~ several mg
- Measurement of universal gravitation is underway
- Future work: reduction of environmental noises and application to calibration of load cell and measurement of cantilever stiffness